



Case studies of air cooled vacuum steam condensers performance improvements

*2013 ACCUG Meeting
October 14-17, 2013*

CASE 1 - ACC CLEANING SYSTEM & FAN OPTIMISATION



Situation ACC until 2010

- **Cleaning of ACC until 2010 was max. 2 times/year**
- **Simply cleaning in a fixed period without looking at loss of performance**

Reduction in production of steam due to poor condensation

Temperature up to 25°C	Percentage reduction in steam compared to maximum design temperature at 25°C	Reduction of steam compared to maximum steam flow at 25°C	Reduction in waste incinerated with steam flow at 25°C
25 to 28°C	Up to 6.23%	Up to 2.65 kg/s	Up to 3.24 t/h
29 to 32°C	Up to 14.54%	Up to 6.18 kg/s	Up to 7.67 t/h
33 to 36°C	Up to 22.86%	Up to 9.70 kg/s	Up to 12.02 t/h

Revenue losses through loss in electricity production

Example of calculations for loss of revenue (minimum case)

30 days temperatures over 25 °C approx. 10 h/day

300 h x ca. 5t/h loss of waste throughput

=

1500 Tons less throughput

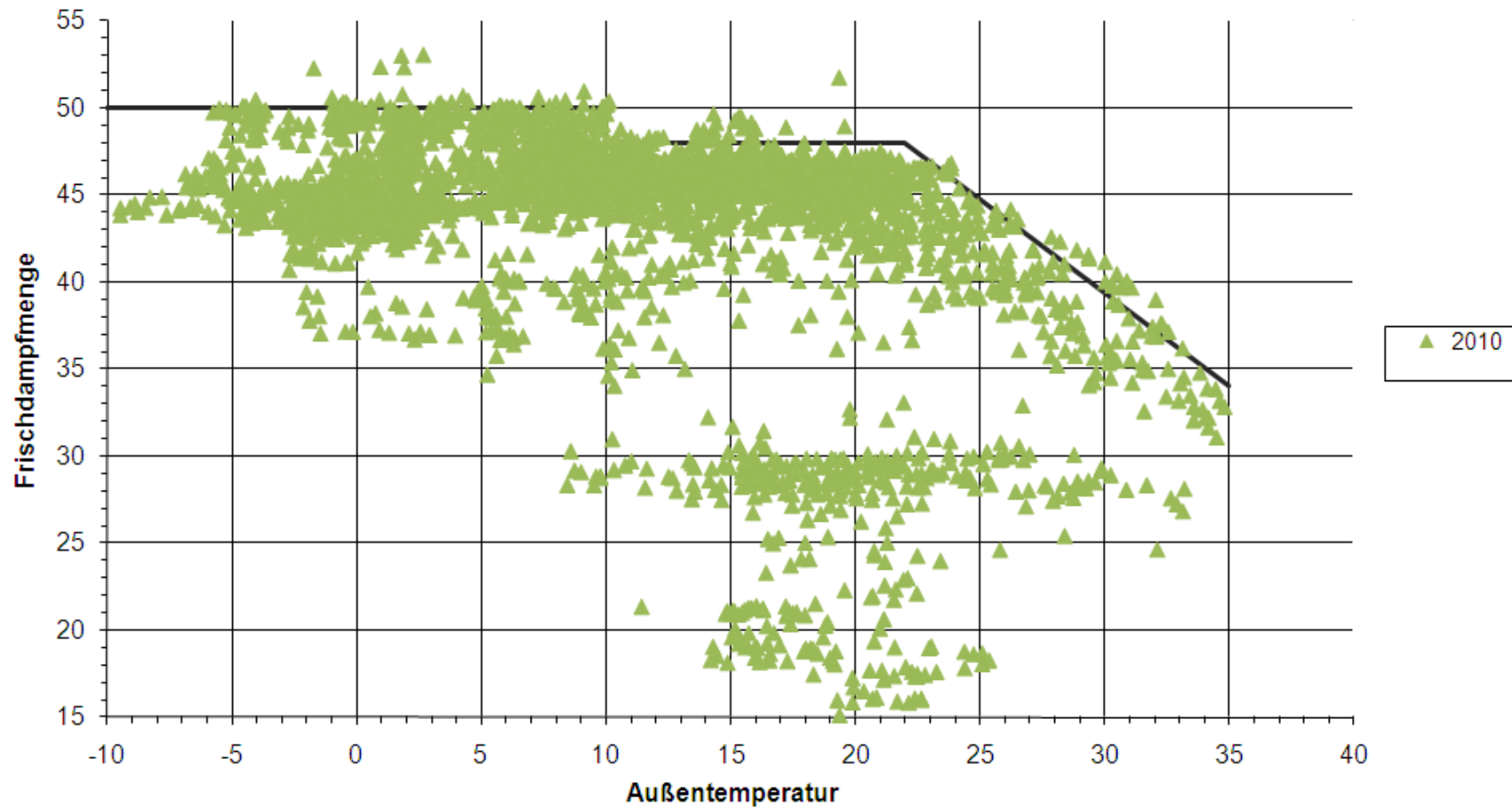
+

5kg/s steam = approx. 5MW * 300h

=

1500 MWh less electricity throughput per year

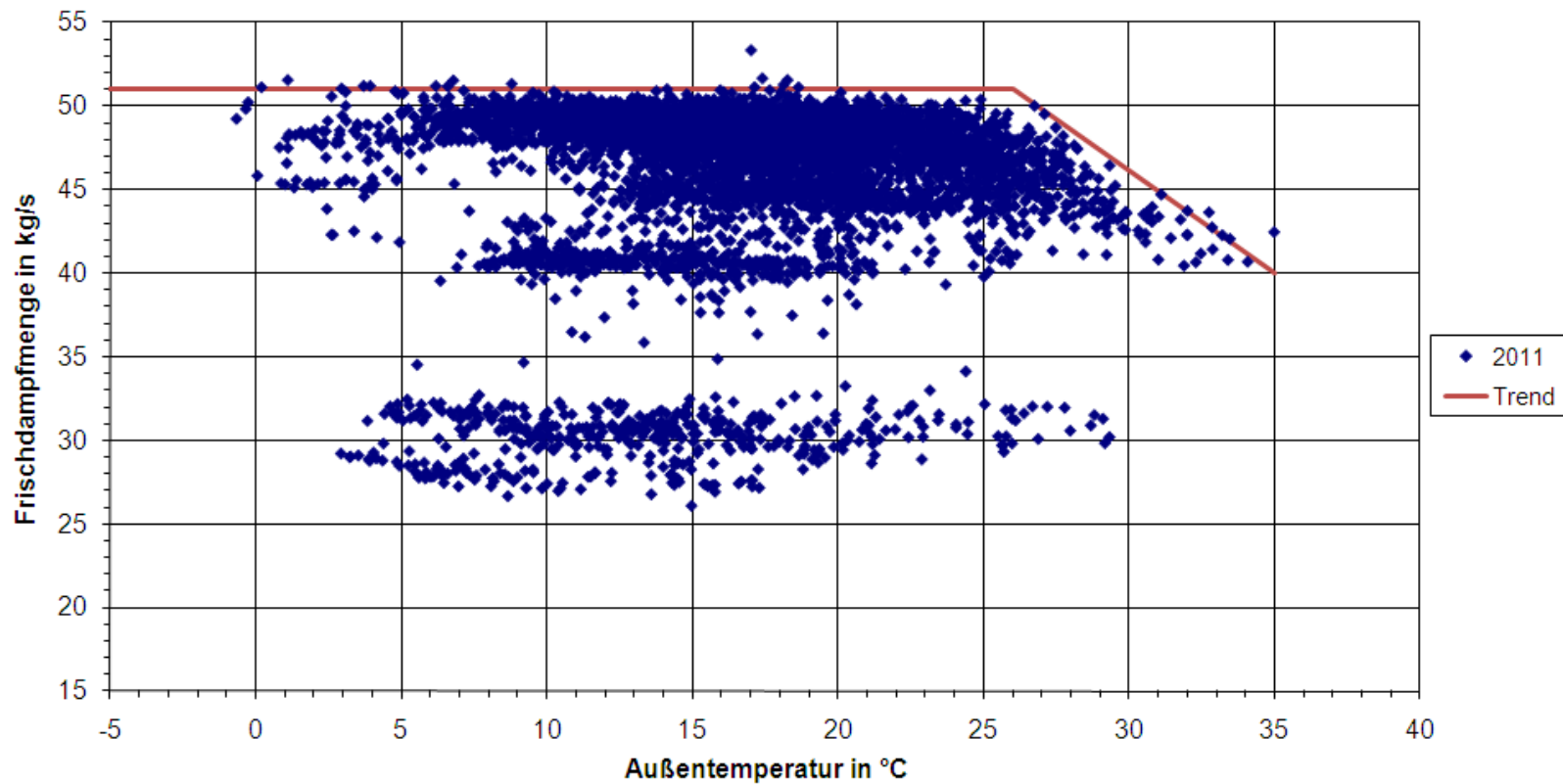
Evaluation 2010



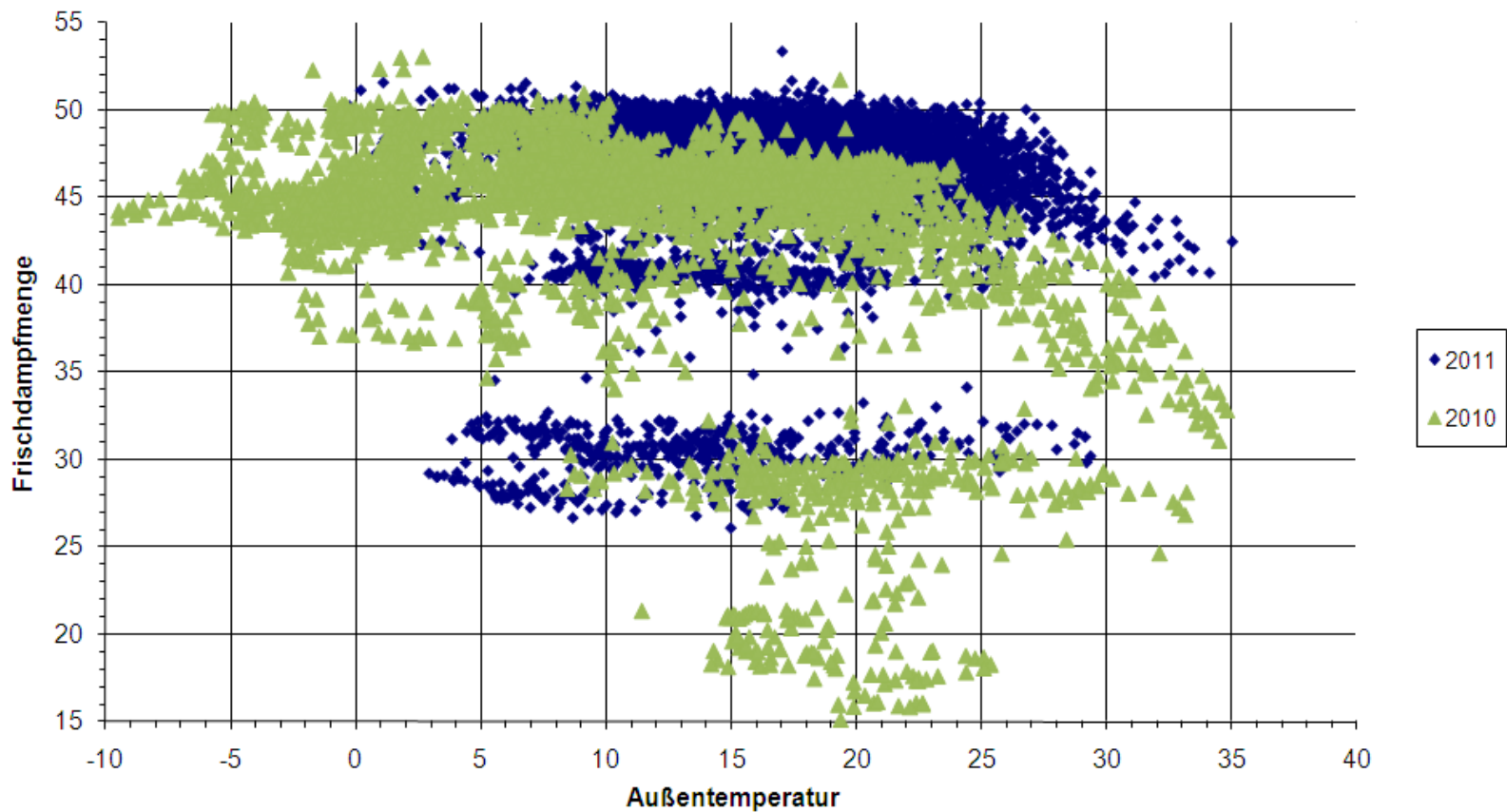
Situation in 2011

- **1. Cleaning after winter**
- **2. Cleaning after middle pollen phase**
- **3. Cleaning after end of pollen phase**
- **4. + 5. Cleaning in summer**
- **6. Cleaning in Autumn**
- **7. Cleaning before winter**

Evaluation 2011



Comparison between 2010 & 2011



Challenges for 2012

Mission

- **Faster cleaning**
- **Improved efficiency of fan system → more condensation capacity**

Optimization Winter 2011-2012

Original cleaning system

- Flexible hoses with check valves
- HP unit working pressure = 110 bar
- Number of nozzles for cleaning head = 12
- Incorrect position of nozzles versus bundle and nozzle orientation
- No option to rinse plenum chamber (after cleaning) and bottom rows bundle



Optimization Winter 2011-2012

Improved cleaning system

- Removed check valves and installed ball valves
- New high capacity HP unit working pressure incl. Soft-start = 120 bar
- Number of nozzles for cleaning head = 24
- Correct position of nozzles versus bundle and nozzle orientation
- Tube union Tee for HP gun (rinsing off dirt inside plenum and lower rows of bundle)



Optimization Winter 2011-2012



Optimization Winter 2011-2012



Improvement made by modification cleaning machine

- **Before it took 7 hours to clean 1 side (2 fans) and with new optimized system is took only 3 hours for 1 side**
 - Only 1 passage for cleaning head now versus 2 passages before due to higher pressure and better positioning & orientation of the nozzles
 - Cleaning head approx. 2 times wider then original cleaning head
- **Better cleaning result per cleaning**
- **4 cleaning sessions per year (versus 7 in 2011) due to improved cleaning results**

Optimization Winter 2011-2012

Original fan system

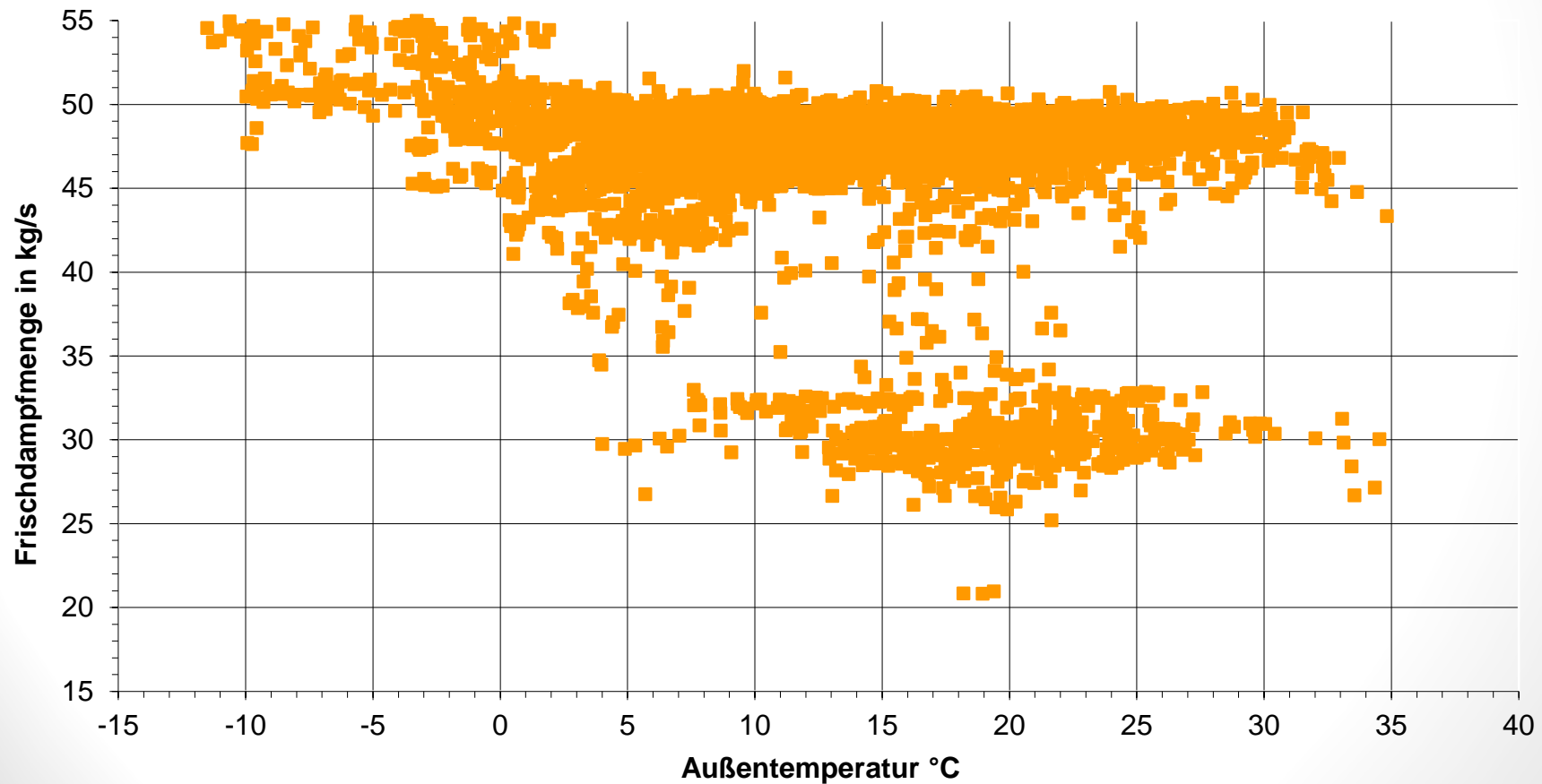
- Margin over on motors
- Fairly low pressure capacity of existing fans → as soon as ΔP increases due to wind or external fouling, fans was starting to cavitate/stall

New Situation fan system

- Increase by 15% in fan RPM by increasing frequency of VSD's from 50 Hz to 57.5 Hz.
- Decrease in pitch angle from 19.1° to 14.0°
- Improved airflow of minimum 8-10% versus original situation
- Improved pressure capacity of fan to cope with wind and especially external fouling

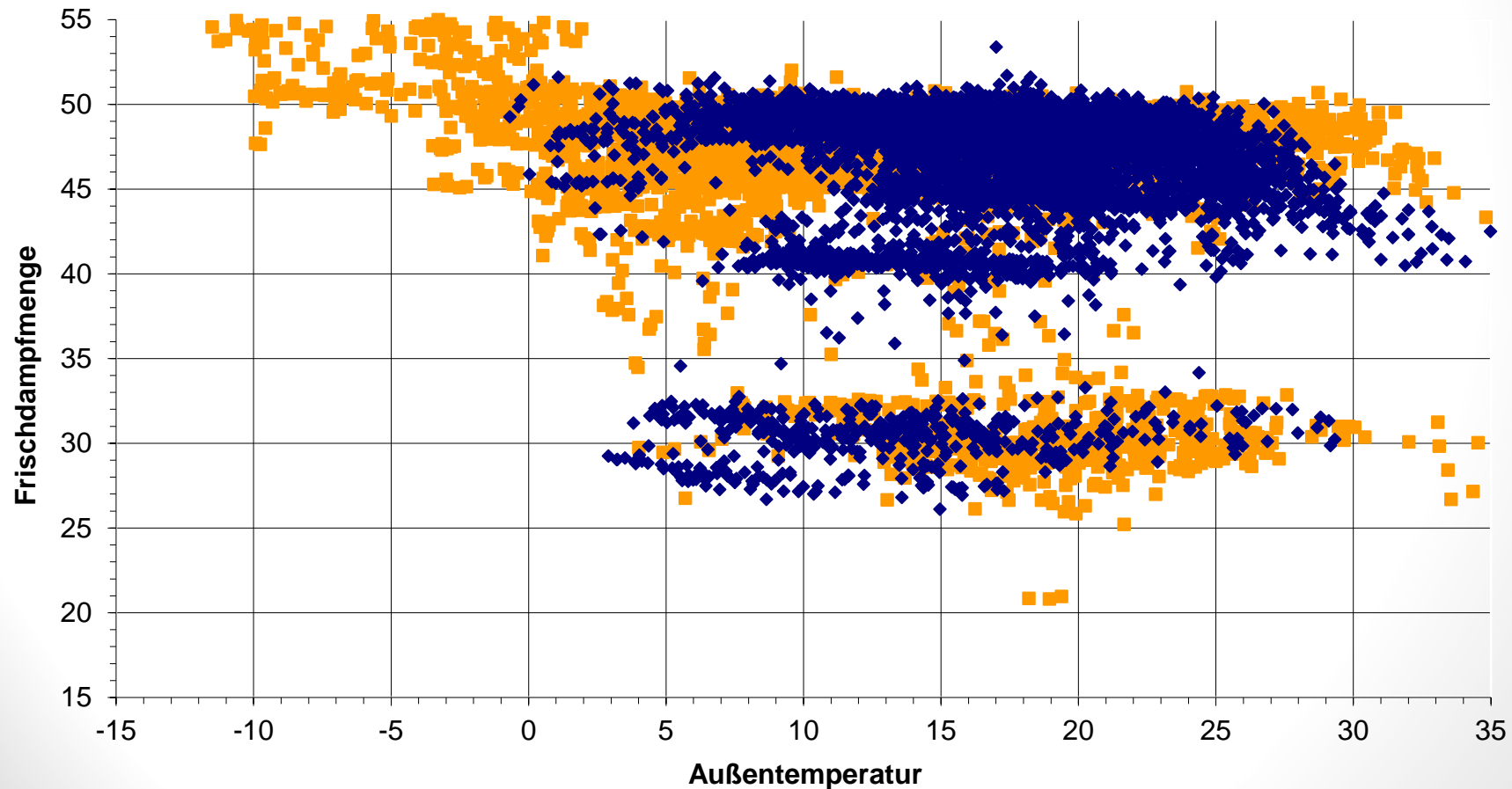
Evaluation 2012

LUKO-Auswertung MHKW 2012



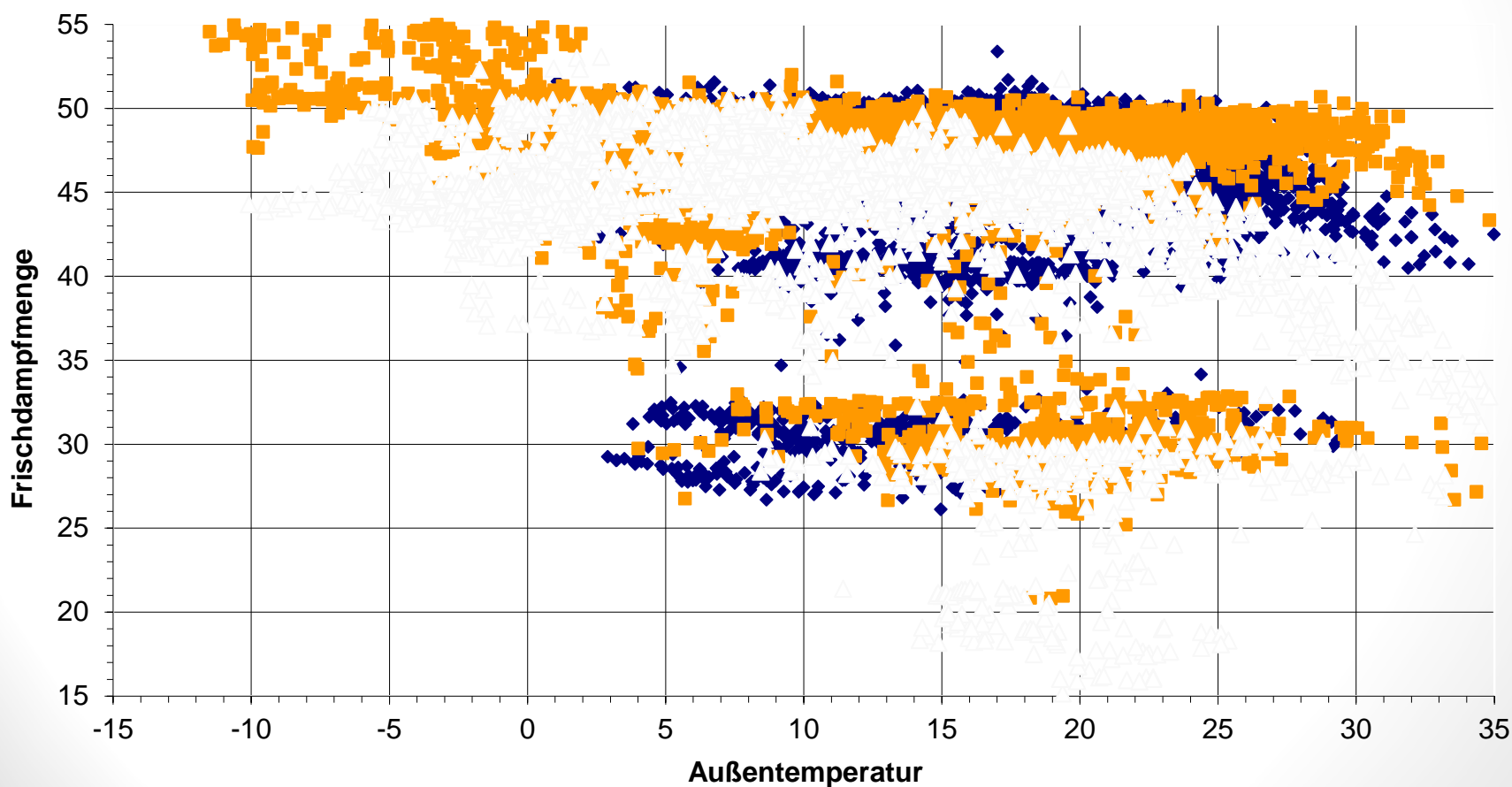
Comparison between 2011 & 2012

LUKO-Auswertung MHKW
2011 zu 2012

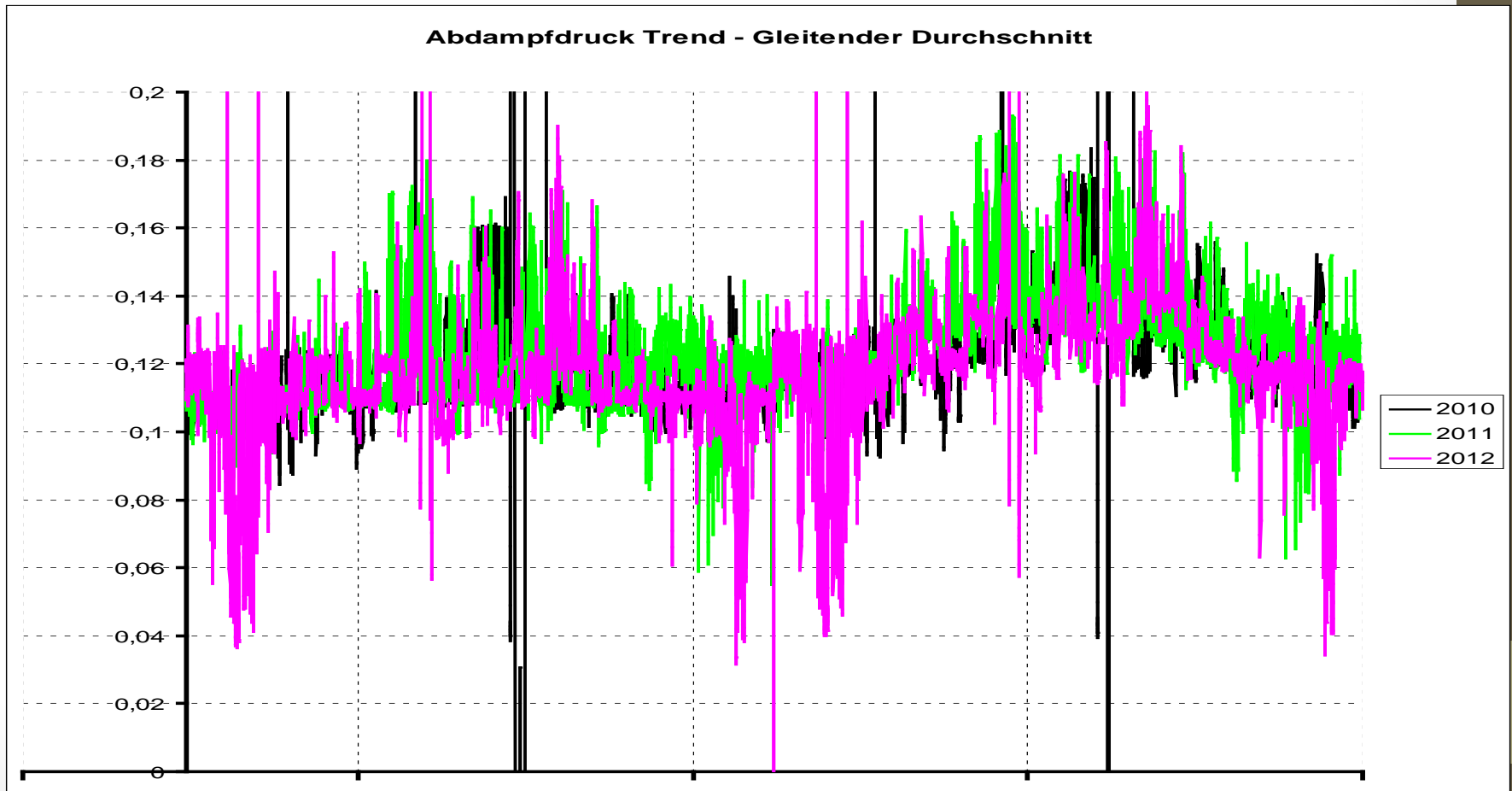


Evaluation 2010 / 2011 / 2012

LUKO-Auswertung MHKW 2010 zu 2011 und 2012



Evaluation 2010 / 2011 / 2012 (Back pressure)



Consequence of improvement



CASE 2 – INSTALLATION OF VARIABLE SPEED DRIVES

Existing situation

- No variable speed drives
- No room for VSD in sub-station (MCC)
- Pitch angle : 19°
- Fan tip speed : 30.9 m/s
- Motor power : 162 amps
- Temperature range : -20°C + 30°C

Average measured data

Airflow: 534 m³/s Static pressure: 55 Pa Amperage : 136 Amps

Traditional Solutions

- New fan with more blades or wider blades AND / OR
- New higher rating motor AND / OR
- Increased heat transfer surface

ELFLOW BV solution



- Increased tip speed of fan by 20% (50 to 60 Hz)
- Decreased pitch angle from 19° to 16°
- Fully loaded motor to 158 Amps

ELFLOW BV SOLUTION ADVANTAGES

New IP20 VSD with IP66 casing inside each cell instead of in sub station

Savings

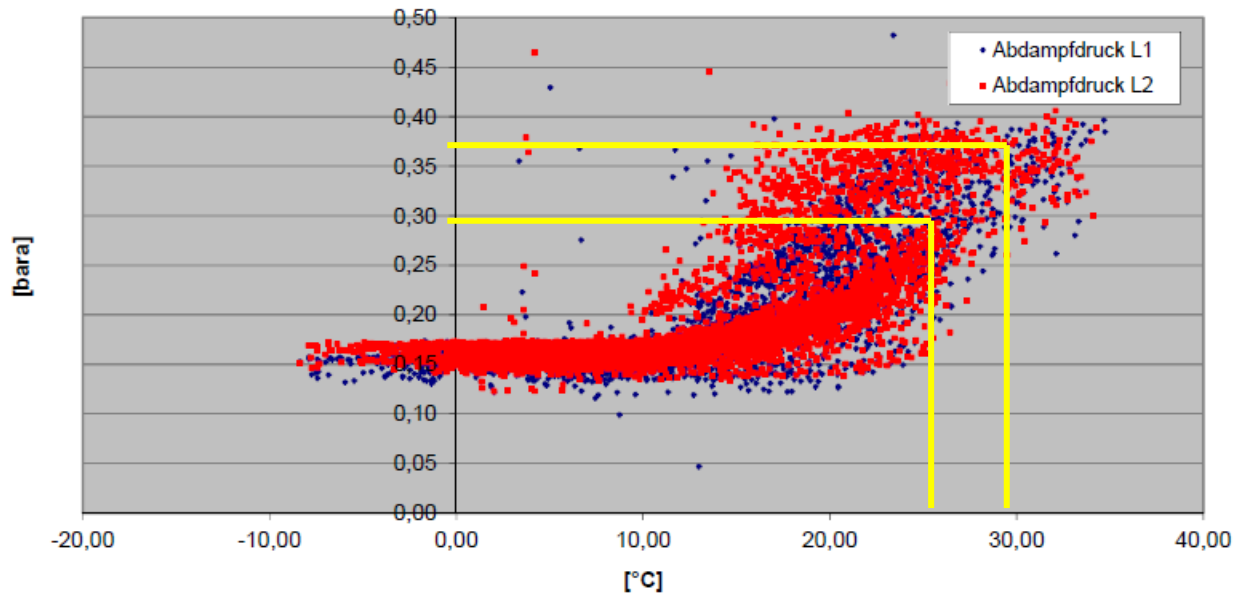
- No new sub-station
- No new fans, motors or bundles
- No costly DU/DT filter (close to motor)
- No costly Sinus filter (close to motor)
- No extra cooling as fan is cooling VSD
- No cabling from sub-station to motor (if VSD is installed in sub station)

ELFLOW BV SOLUTION ADVANTAGES

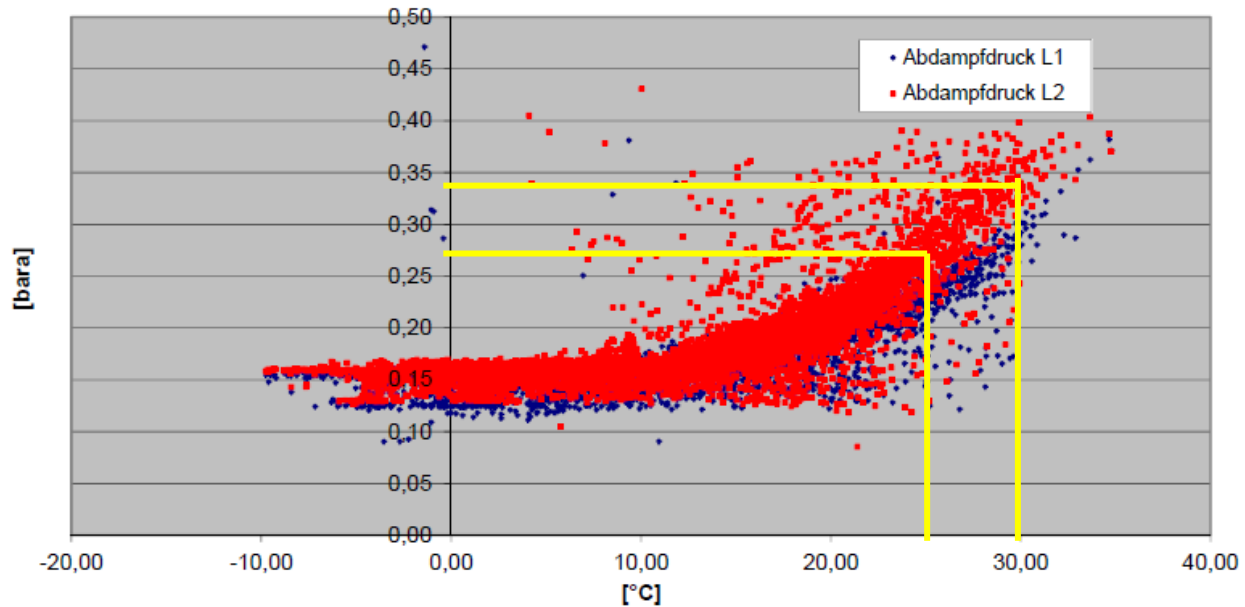
Advantages

- No need anymore for extra capacity on motor → Fans will turn slower in winter as cooling is not an issue at -20°C!!!!!! This power can be utilized to increase airflow during summer operation
- Lower noise level during night/cooler period (when it is cooler and when noise limitations are stricter)
- Decreased total power consumption required by ACC during a year as regulation is optimized by VSD
- Strongly reduced load on fan blades, gearboxes and motor when running at lower RPM
- No wind-milling (fans will only run slower)
- Reduced maintenance costs of gearbox and fans (no start-stop)
- Ability to increase fan RPM by approx. 20% (50 to 60 Hz or 60 to 70 Hz) and therefore efficiency and pressure capacity
- Increased pressure capacity of fan by approx. 40%
- Increased static efficiency of fans by 6%
- Increased airflow of fans by 14% versus original situation

Abdampfdruck in Abhängigkeit der Aussentemperatur 2008



Abdampfdruck in Abhängigkeit der Aussentemperatur 2011



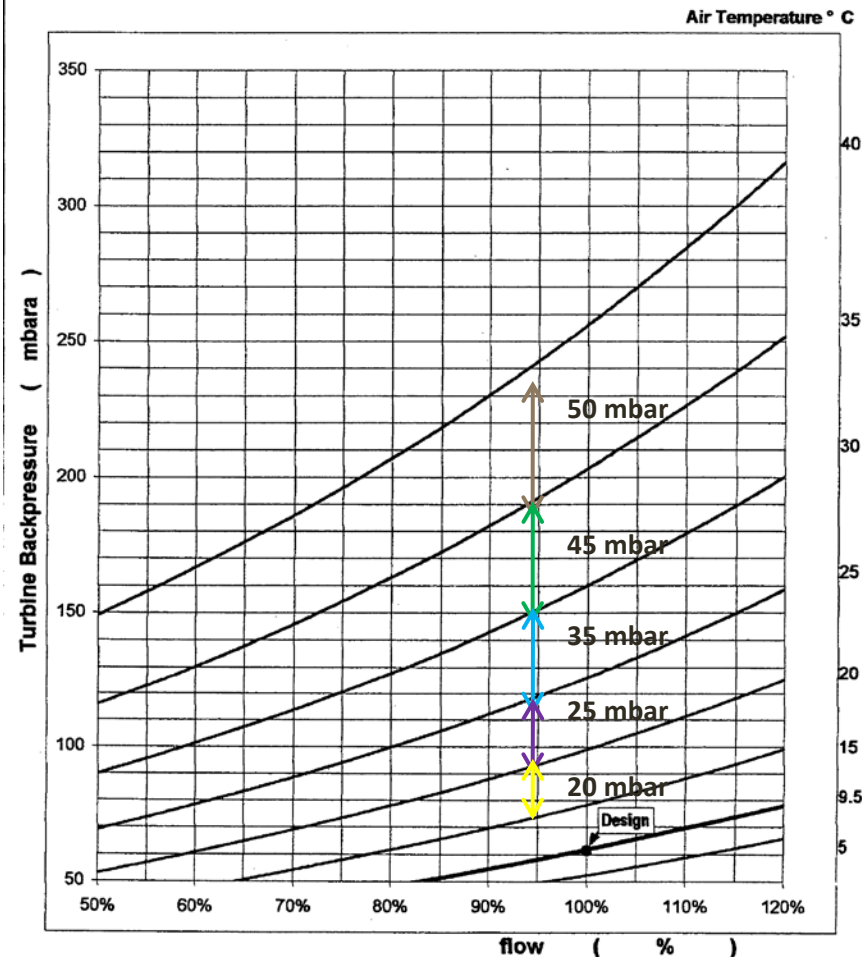
CASE 3 – ADIABATIC COOLING FOR ACC IMPROVEMENT



Adiabatic cooling

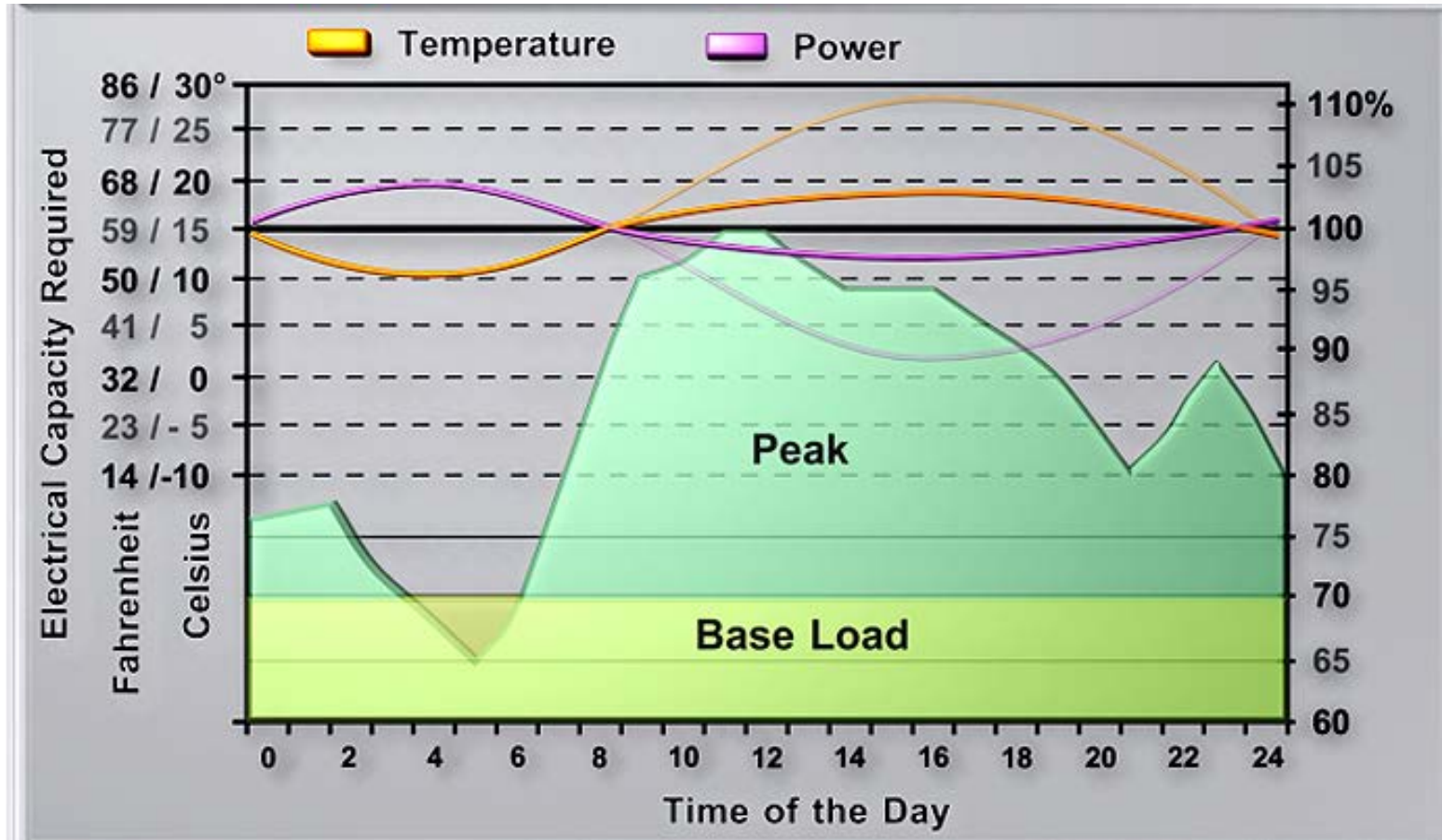
DESIGN CONDITIONS

Steam Flow	DG	100.00	kg/s	Steam Dryness	xG	0.920	kg/kg
Back Pressure	PG	61.00	mbara				
Air Temperature	tLG	9.5	C	Barom. Pressure	bG	978.85	mbara



Typical ACC performance curve (Alstom turbine) illustrating improvement in vacuum with increments of 5°C in air temperature (at same steam flow rate)

Price of electricity varies with air temp.



Reduce ambient air temperature

Proper chilling system depends on



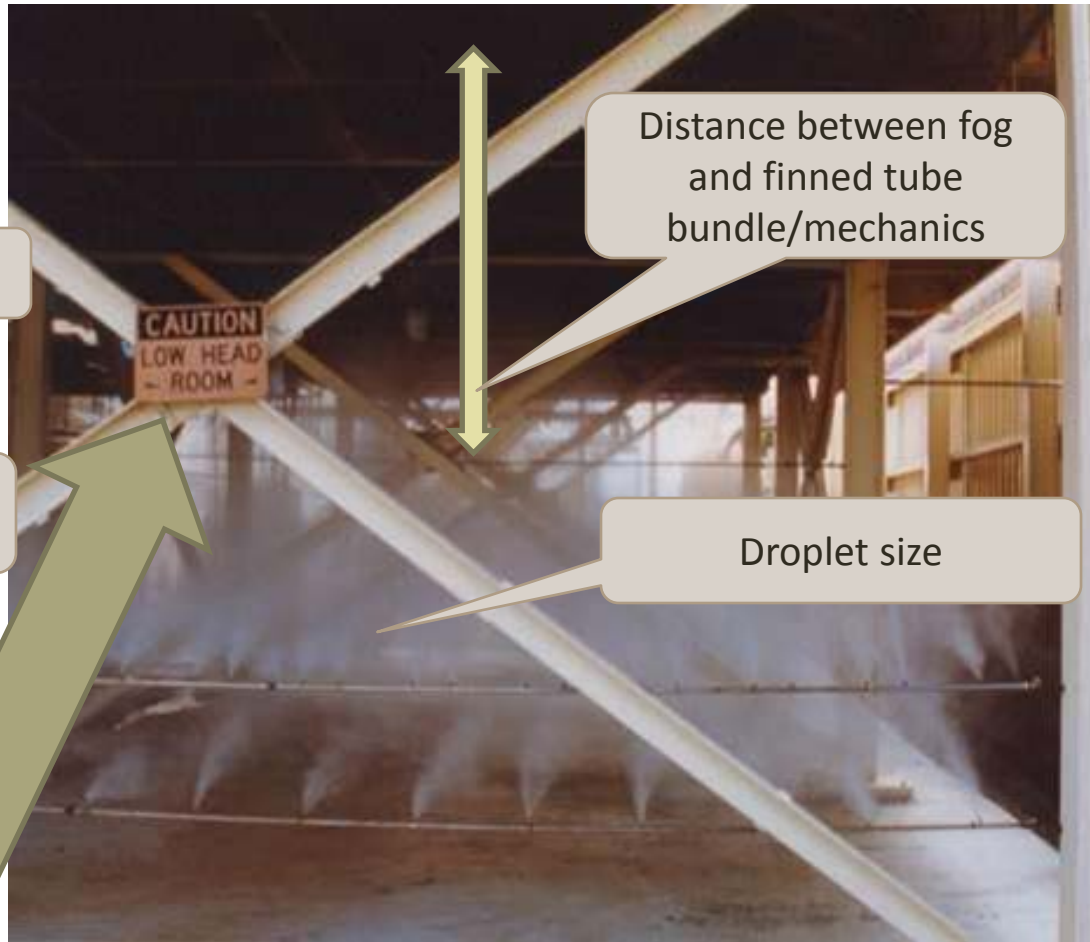
25 °C

Climate conditions

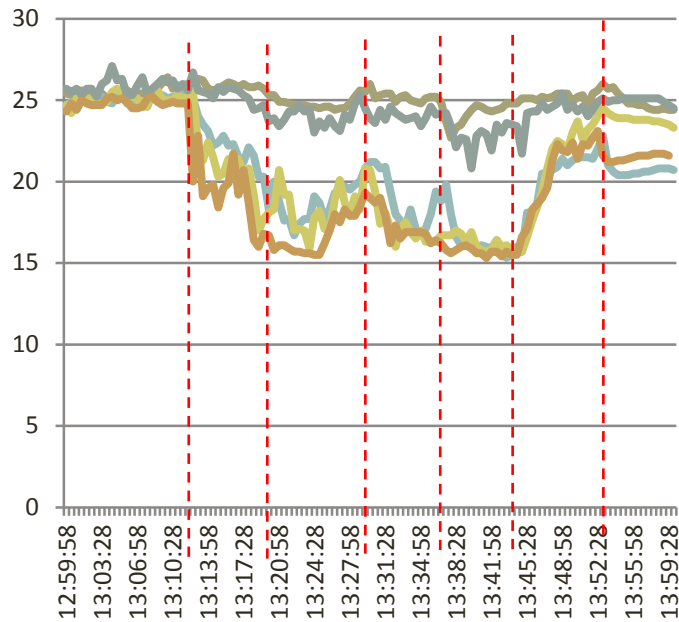
Face velocity

Distance between fog
and finned tube
bundle/mechanics

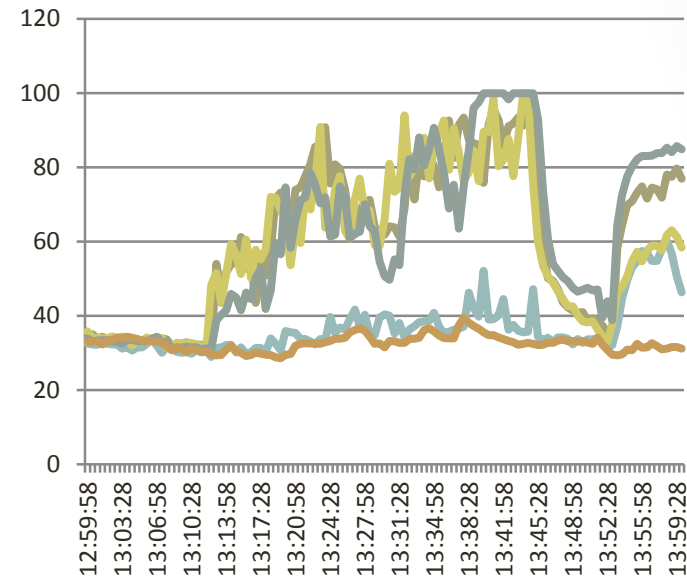
Droplet size



Possible Reduction in ambient air temperatures using COOLINGMIST™ system



Temp MGR015
Temp MGR016
Temp MGR017
Temp MGR018
Temp MGR019



RH MGR019
RH MGR018
RH MGR017
RH MGR016
RH MGR015

