

Winter, ACC's, and Lost MW's

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Winter, ACC's, and Lost MW's

- Winter back pressures are too high
- The resulting MW loss can be significant (up to 5% of STG MW's)
- The loss depends on “End Loading”
- The high backpressures are due to high air in-leakage, metal wastage concerns, freezing concerns
- Heatrate.Com and Power Software are developing instrument and control mods to minimize these losses.

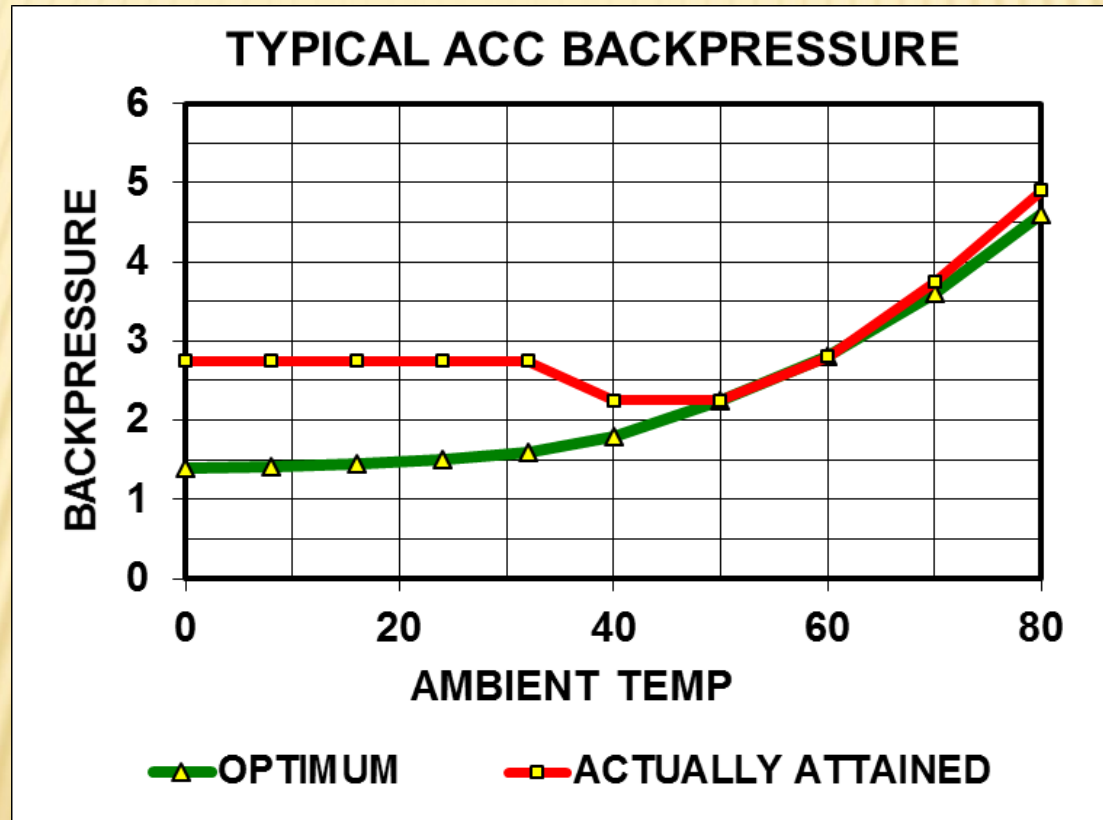
Winter, ACC's, and Lost MW's

- **Jim Koch, Performance Consultant,
www.heatrate.com, 30+ years in Power
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- **Chris Haynes, Power Software Associates
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- **Dr. Melanie Derby, Kansas State University
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Interest in Heat Transfer & Condensing**

Summer Backpressure

Winter Losses due to High Backpressure

Many steam turbines operate at higher than optimum backpressures during the winter



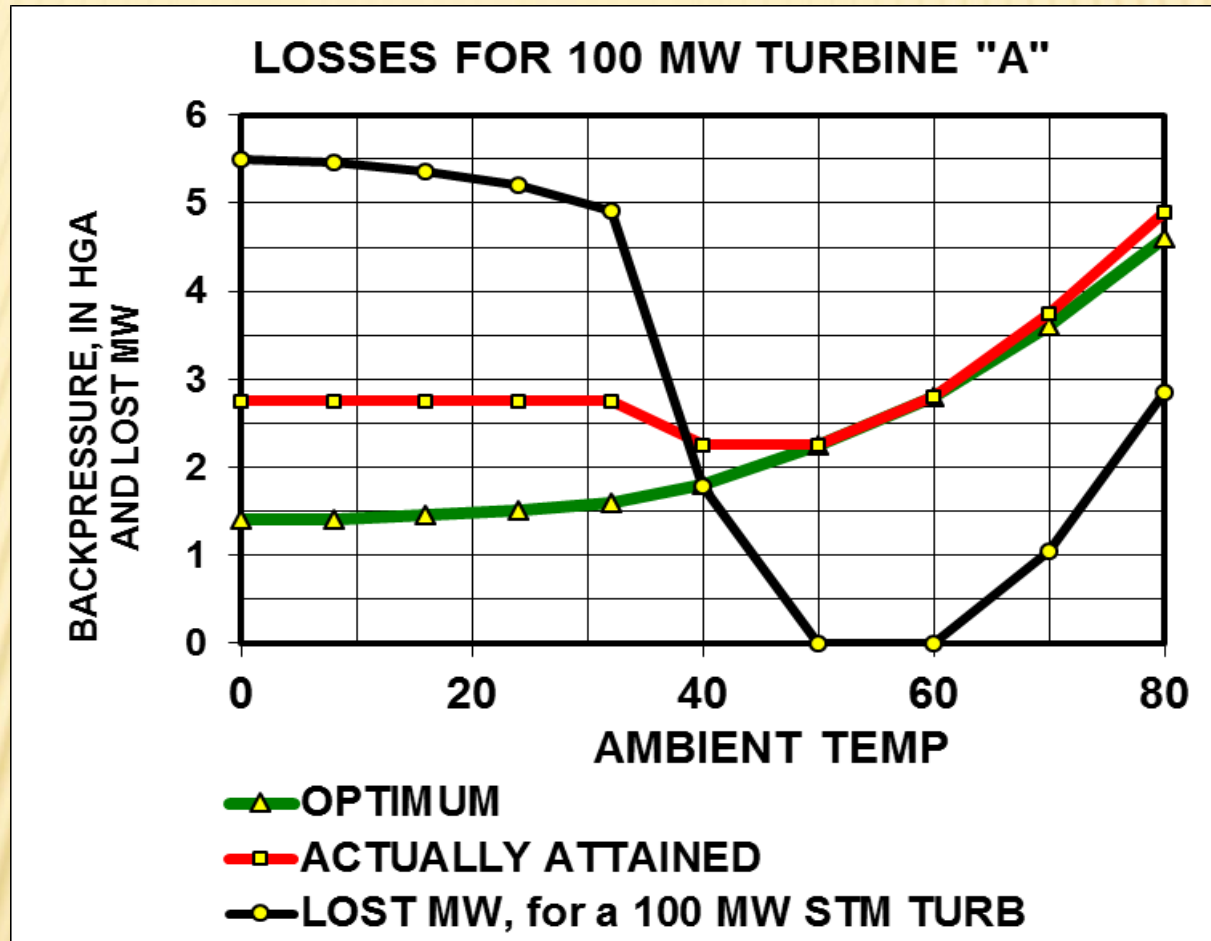
Winter Losses due to Backpressure

- Owners know that high backpressure is bad
- This is usually only a summertime concern
- Owners are often unaware of possibility of lost MW's during the winter
- The loss can often be 5% of STG MW's
- For a 100 MW STG, this can be up to \$1.5 million per winter with high power prices
- These losses are unit specific

Rule of Thumb ?

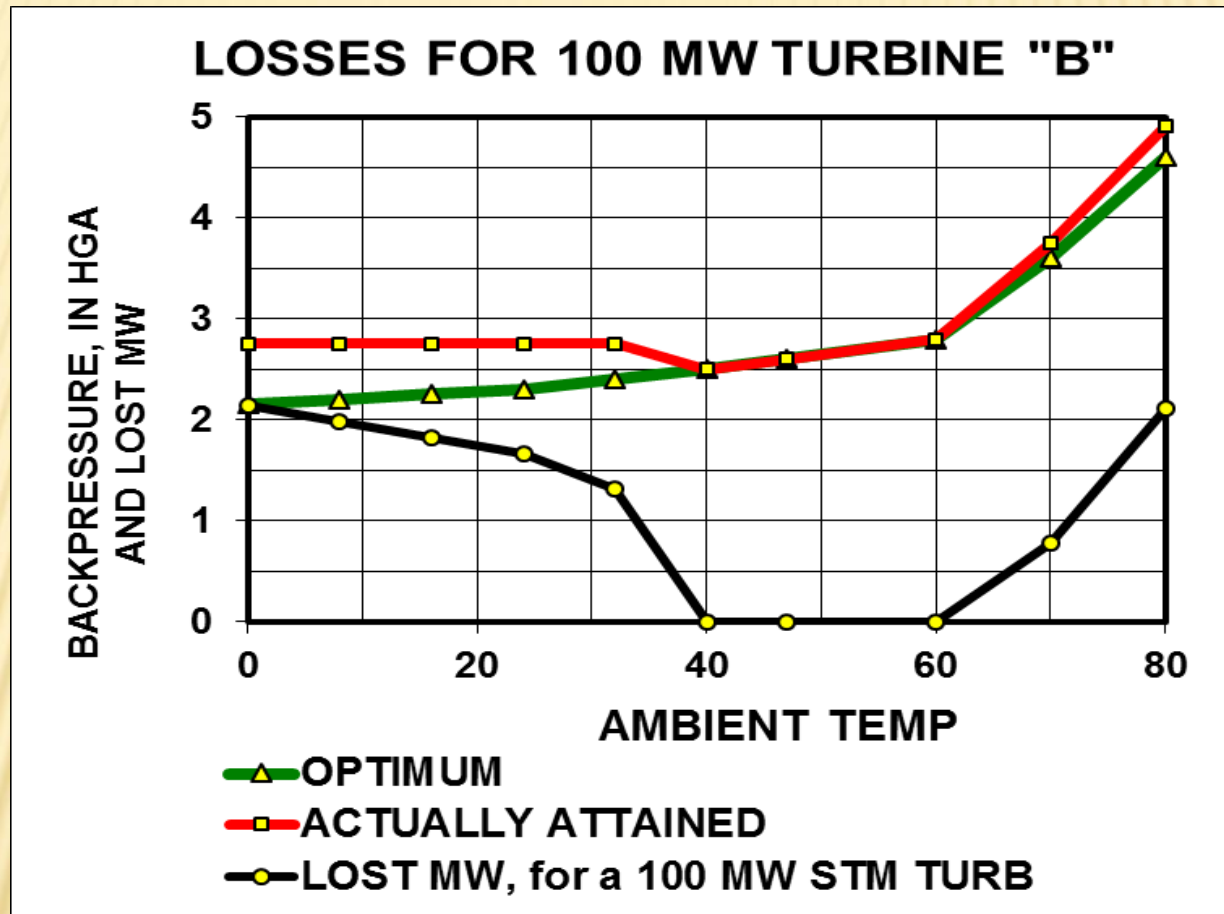
- Years ago, common adage was “1 inch equals 1%”
- This is **INCORRECT ! !**
- The rule of thumb is that there is **NO** rule of thumb
- The relation between back pressure and MW's is **UNIT SPECIFIC**

Turbine A



Up to 5% of output can be lost.

Turbine B



Losses are, at most, 2% of output.

Turbine A vs Turbine B

- Turbine A and Turbine B have very different losses due to high backpressure
- The difference is due to the size of the last stage blade, relative to steam flow
- This is called the “End Loading”

End Loading

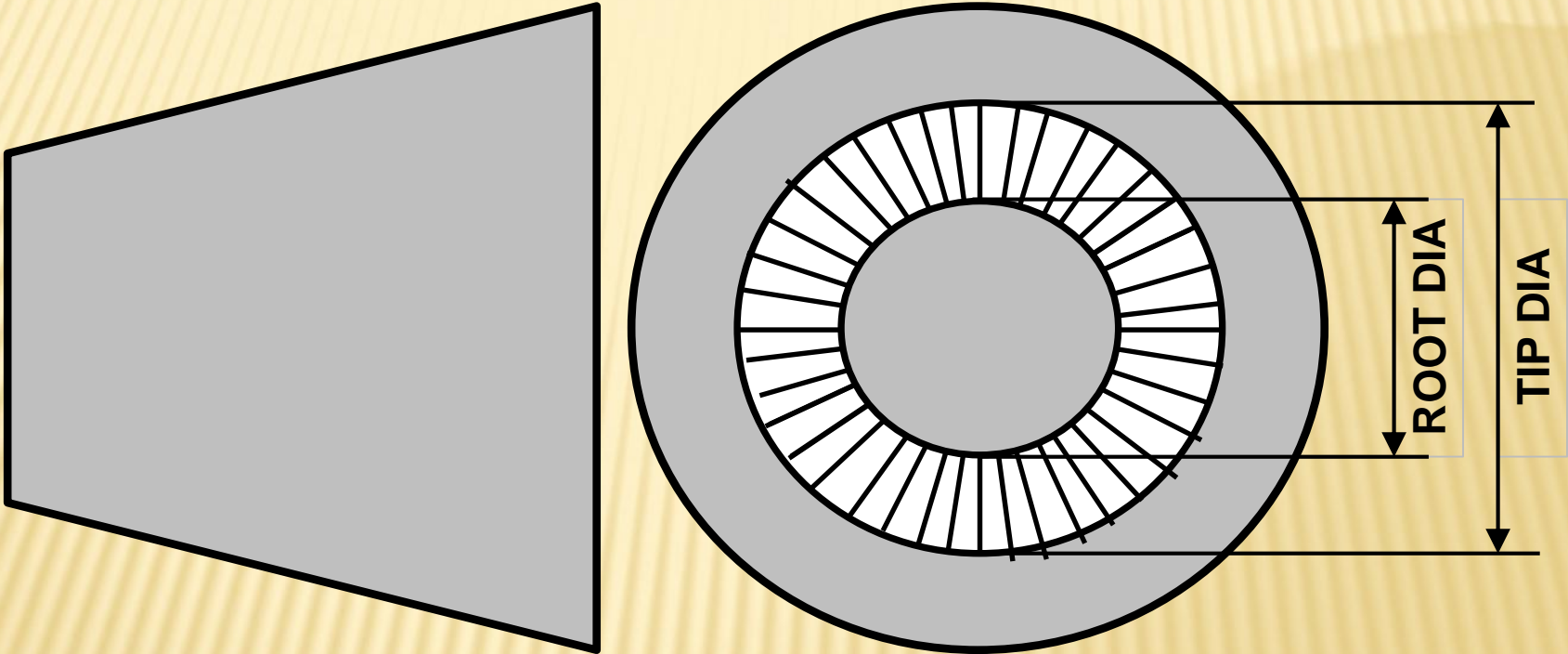
What is “End Loading?”

End Loading = $\frac{\text{Lb/hr of exhaust steam flow}}{\text{(Square feet of exhaust flow area)}}$

Very Small Last Stage: 18,000 lb/hr-sqft

Very Large Last Stage: 6,000 lb/hr-sqft

Exhaust Flow Area



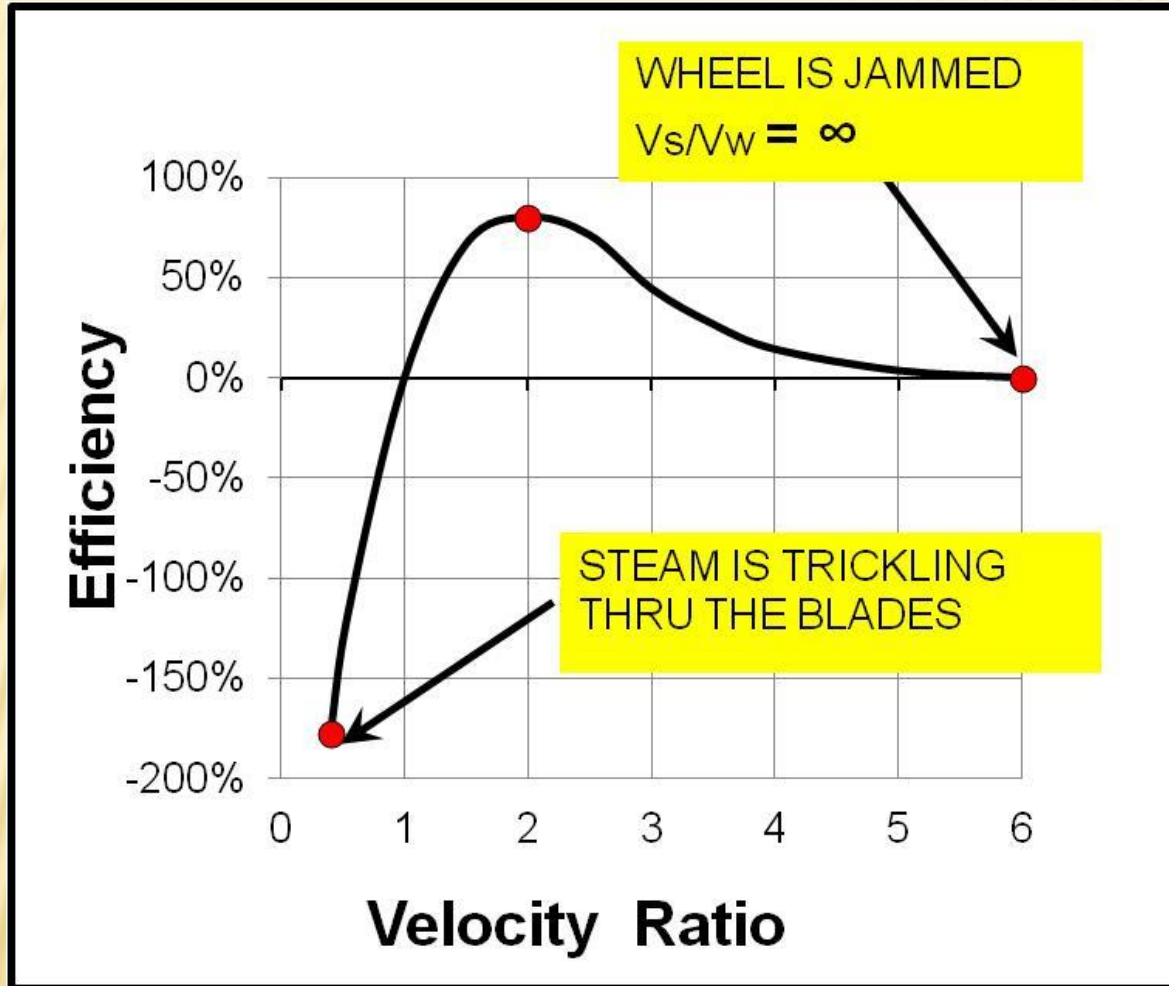
End Loading

Q. Why does End Loading matter?

A. It sets the Blading Efficiency, for any given steam flow and backpressure

- **End Loading determines Steam Velocity**
- **Steam Velocity determines Velocity Ratio**
- **Velocity Ratio determines Blading Efficiency**

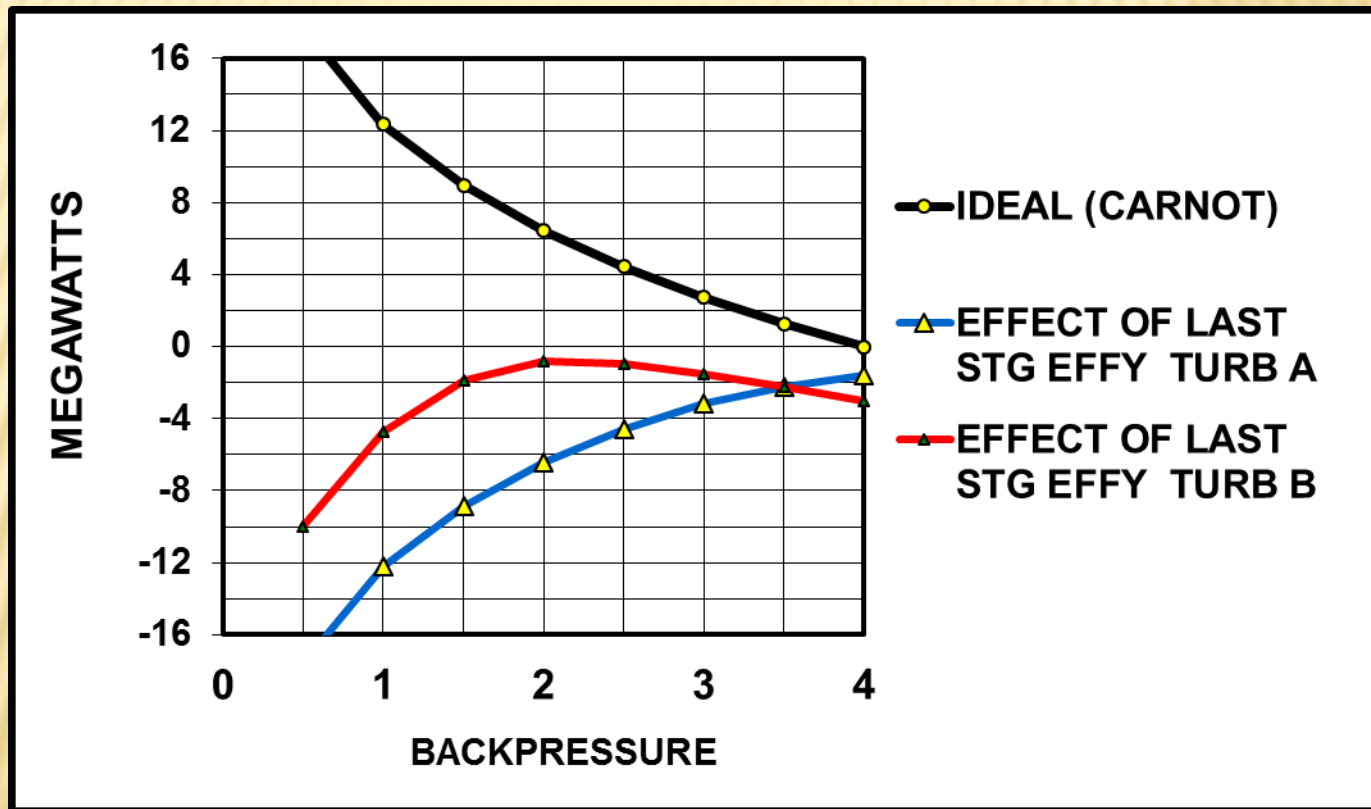
Efficiency Depends on Velocity Ratio



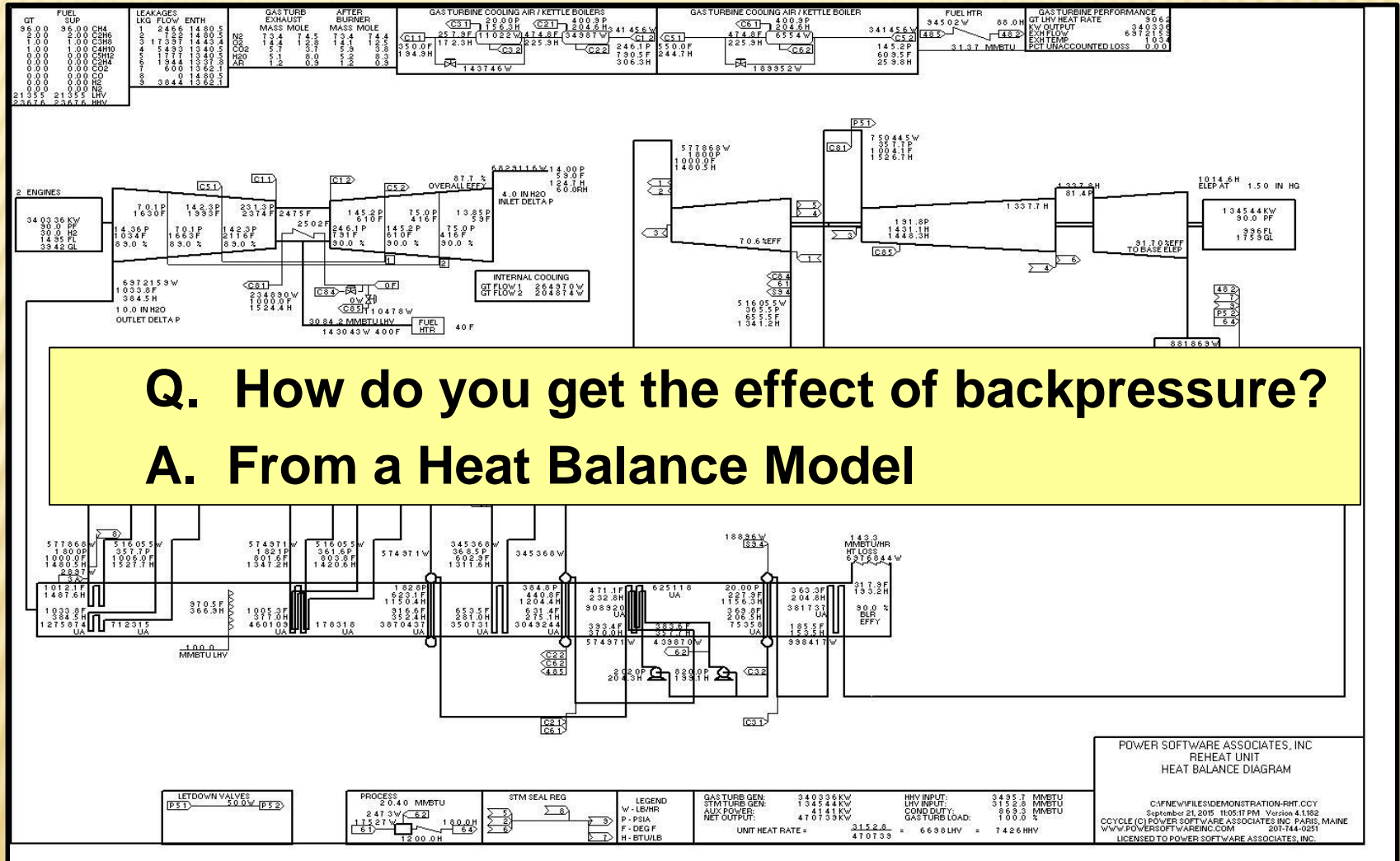
How does Backpressure affect MW's?

Backpressure has two effects:

1. Ideal (Carnot) cycle efficiency changes
2. Last stage blade efficiency changes

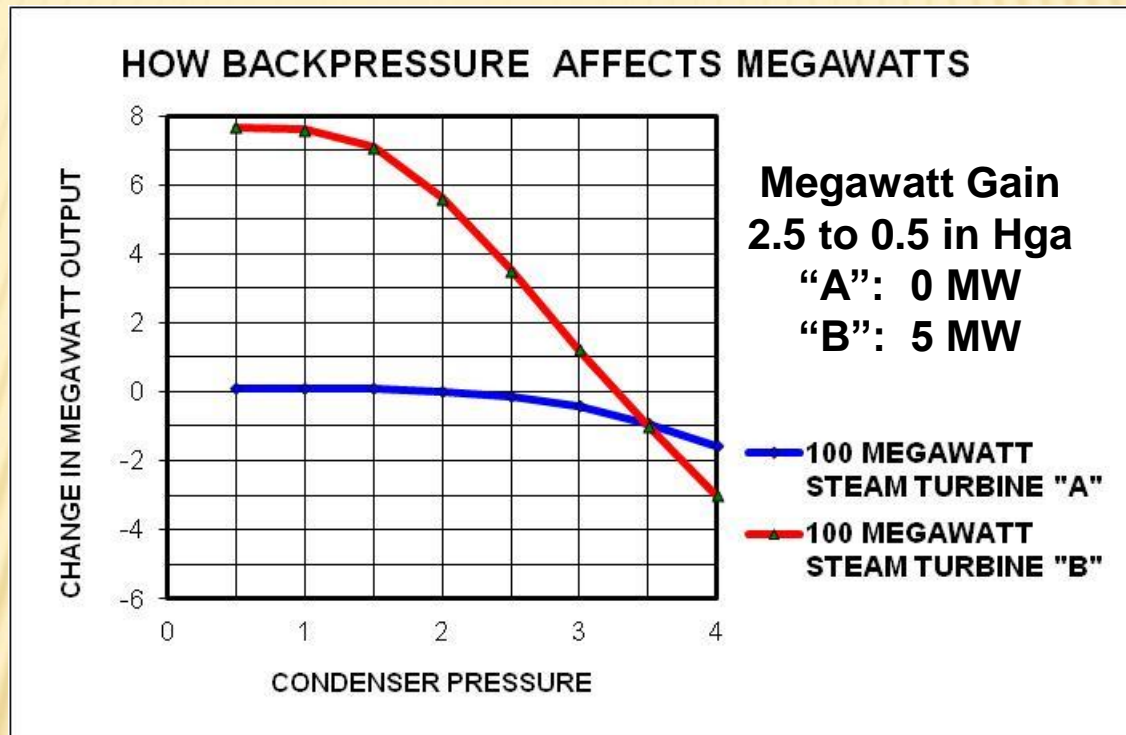


How does Backpressure affect MW's?



How does Backpressure affect MW's?

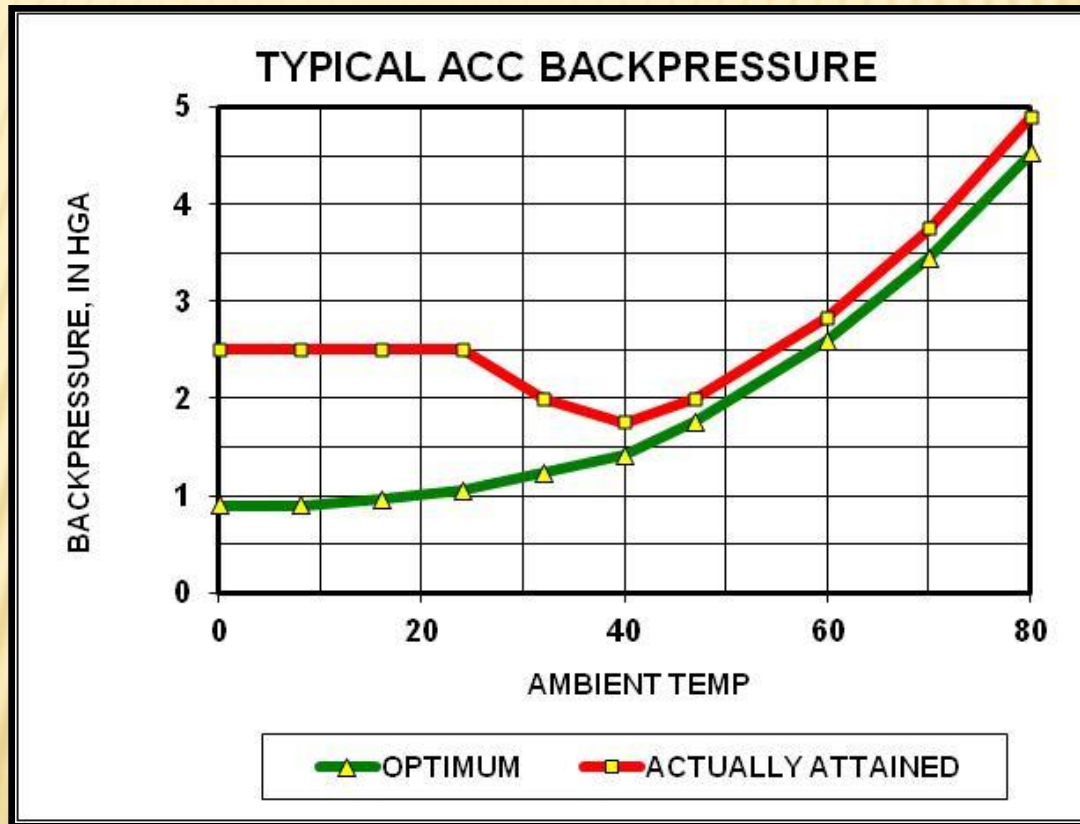
Megawatt losses vary from one unit to another.



A has larger last stage, lower end loading
B has smaller last stage, higher end loading

Winter Time Backpressure

Revisit typical curve of back pressure vs. ambient T
Focus on region between 32 F and 40 F



Backpressure, Dollars, and Freezing

- **The trade-off between backpressure and enhanced freezing protection methods depends on End Loading**
- **In HIGHLY End Loaded Units, there is LITTLE advantage to reducing backpressure**
- **In LIGHTLY End Loaded Units, there is GREAT advantage to reducing backpressure**

Why are Winter Backpressures so high?

1. High air In-Leakage
2. Erosion concerns
3. Freezing concerns

In-Leakage and Winter Backpressure

- **High air in-leakage can increase backpressure. *It also increases danger of freezing.***
- **ACC's are more susceptible to air in-leakage than water cooled units**
- **Air removal pumps have less capacity when backpressure is low**

Backpressure and Erosion concerns

- **“Erosion” includes both conventional erosion and flow accelerated corrosion (FAC).**
- **Both are worsened at LOW backpressure, as velocity and amount of moisture increase**
- **FAC is being mitigated with chemistry control**
- **Progress in corrosion mitigation now makes lower backpressures achievable.**

Need for Better ACC Freeze Protection

- **Conservative freeze protection procedures prevent plants from taking advantage of achievable lower winter back pressure**
- **When “not quite freezing” these guidelines are too restrictive, margins are overly-conservative**

Need for Better ACC Freeze Protection

- **Visited a 125MW STG site with 18 cell ACC**
- **1 condensate drain temperature reading per K-cell**
- **Provided limited information to operators**
- **As a result, Operators are discouraged from being too aggressive in winter, leaving MW's "on the table"**

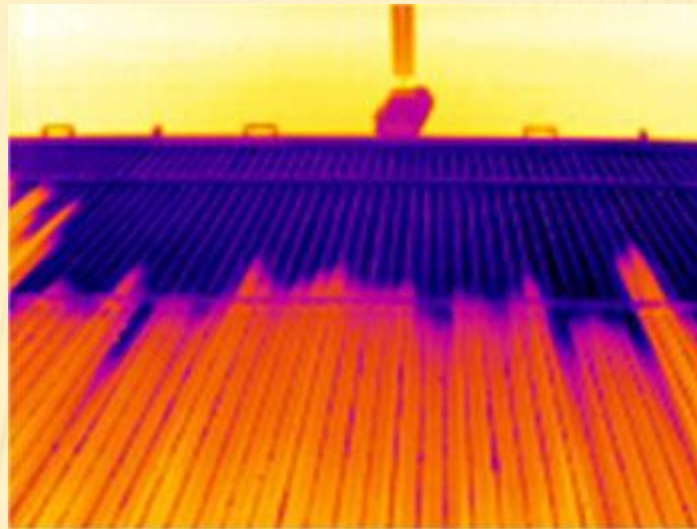
Development of Improved Freeze Protection Instrumentation and Control

- **15 temperature readings are too few**
- **1,000,000 readings are better**

Development of Improved Freeze Protection Instrumentation and Control

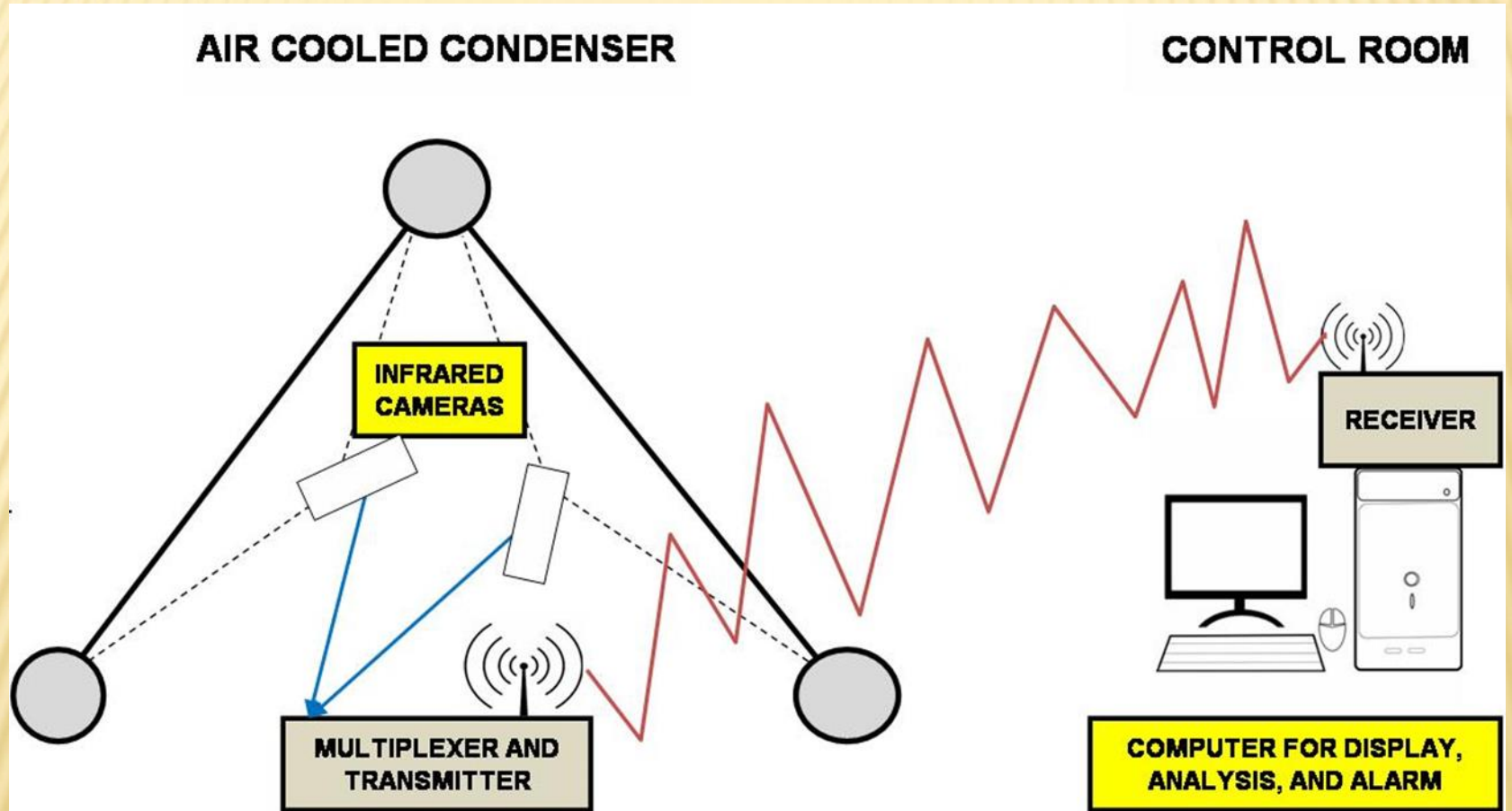
- Development, simulation and field testing of Improved Instrumentation and Controls (patent pending) Technology
- Program to be undertaken by heatrate.com and a selected “Alpha Site”
- Pursue “Beta Site” opportunities
- How to monitor temperature at 1,000,000 locations on the ACC ?

Thermography Measurement, ACC Fan Control System



- IR Thermography will locate cold ACC tubes
- These will be displayed and alarmed in the control room, and digitized to provide control system input
- The Plant DCS control computer will “burp” the air pockets from the condenser

Development of Improved Freeze Protection Instrumentation and Control



Development of Improved Freeze Protection Instrumentation and Control

Tasks:

- Development and field testing of thermography measurement, display and digitizing technology
- Simulation and field testing of burping methods to expel air pockets to prevent freezing
- The “Burp-o-matic” (*patent pending*)

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