## TURNING VANE WIND TUNNEL TEST RESULTS

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Presented at the:

2018 ACCUG Conference Colorado Springs, Colorado October 10, 2018 AIR COOLED CONDENSERS EXPERIENCE FAN VIBRATIONS RELATED TO AMBIENT CROSSWINDS. THE FAN VIBRATIONS CAN RESULT IN COSTLY AND POTENTIALLY DANGEROUS FAN BLADE BREAKAGE.

THE AXIAL WIND-TURNING VANE DESCRIBED IN THIS PRESENTATION IS DESIGNED TO ELIMINATE WIND-INDUCED AXIAL FAN BLADE BREAKAGE.



# PREVIOUS ACCUG WIND TURNING VANE PRESENTATIONS:

Cuerdon, Martin J., P.E., Solving ACC Axial Fan Wind Related Problems, http://acc-usersgroup.org/wp-content/uploads/2013/10/18-Cuerdon.ACCUG-Las-Vegas-Presentation-2013.pdf

Cuerdon, Martin J., P.E., **Axial Fan Wind Turning Vane Scale Model Test Results**, <u>http://acc-usersgroup.org/wp-content/uploads/2014/10/Axial-Fan-Wind-</u> <u>Turning-Vane-Scale-Model-Test-Results.Martin-Cuerdon-Advanced-Analytical-</u> <u>Solutions.pdf</u>

Copies of these presentation slides can be found on the ACCUG website,

http://acc-usersgroup.org/presentations/





#### ORIGINAL TEST APPARATUS PROVED AXIAL WIND TURNING VANE CONCEPT

Previous test results demonstrated the axial fan output flow distribution was;

- 1. Axisymetric bimodal (i.e., 2 Humped Camel shape) with no crosswnd,
- 2. Distorted axisymetric bimodal with a crosswind,
- 3. Significantly less distorted axisymmetric bimodal with a crosswind when the Axial Turning Vane was employed.



## ORIGINAL APPARATUS USED TO PROVE CONCEPT NOT PRECISE ENOUGH TO ANSWER <u>QUANTIFICATION</u> QUESTIONS

- How much does the Axial Wind Turning Vane obstruct air flow when there are no crosswinds?
- Is there a break-even point where the axial fan air flow actually improves with the use of the Axial Wind Turning Vane?
- Can the Axial Fan aerodynamic imbalance be measured?
- How much does the Axial Wind Turning Vane reduce the Axial Fan aerodynamic imbalance?



#### ORIGINAL FAN = 80 MM DIAMETER, 2 VANES NEW FAN = 200 MM DIAMETER, 7 VANES





	Original Test Apparatus	Revised Test Apparatus
Fan Diameter	30mm	200mm
Measurement Interval	0.25"	.0625"
Measurements per Traverse	13	129
Crosswind Condition	Turbulent	Laminar
<b>Crosswind Measurement</b>	No	Yes
Crosswind Control	No	Yes
Ambient Air Temp Measurement	No	Yes
Ambient Air Pressure Measurement	No	Yes





## REVISED AXIAL TURNING VANE TEST WIND TUNNEL

- 10 foot long
- 2 foot x 2 foot Inlet
- 1 foot x 1 foot observation section
- 4 speed furnace fan
- guillotine outlet damper (Not Shown)

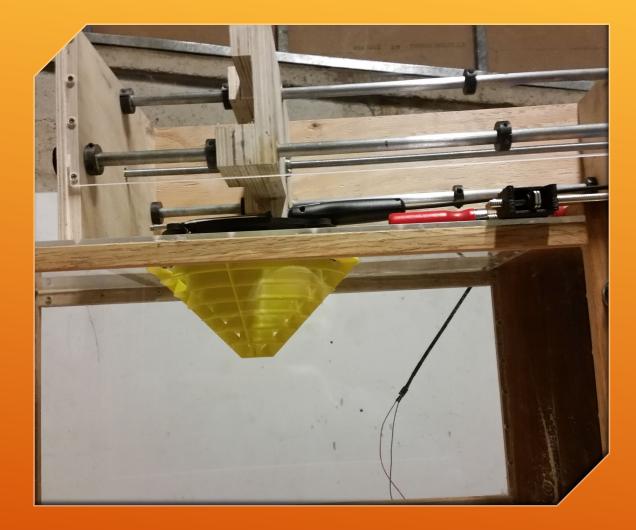
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NEW TEST APPARATUS W/O AXIAL WIND TURNING VANE

Revised Test Apparatus Axial Fan mounted on Wind Tunnel measuring section. Axial Fan output air flow measured a with a hot wire Anemometer mounted on a sliding support indexed by a 3/8 '–16 threaded rod.





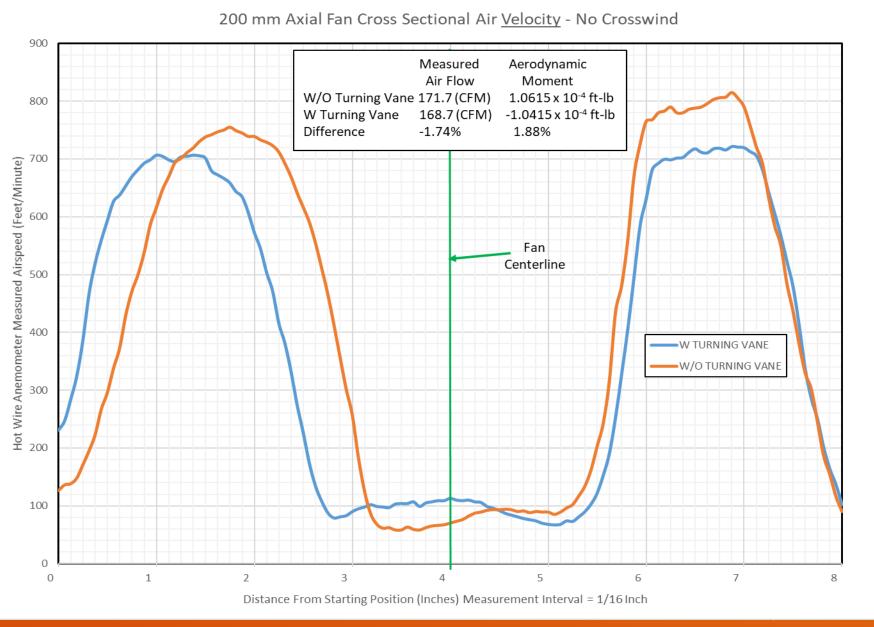
#### REVISED TEST APPARATUS WITH AXIAL TURNING VANE INSTALLED

Original number of measurement points per traverse = 13, interval =  $\frac{1}{4}$ ", traverse length = 3". Measurements per traverse = 1300 (100 x 13).

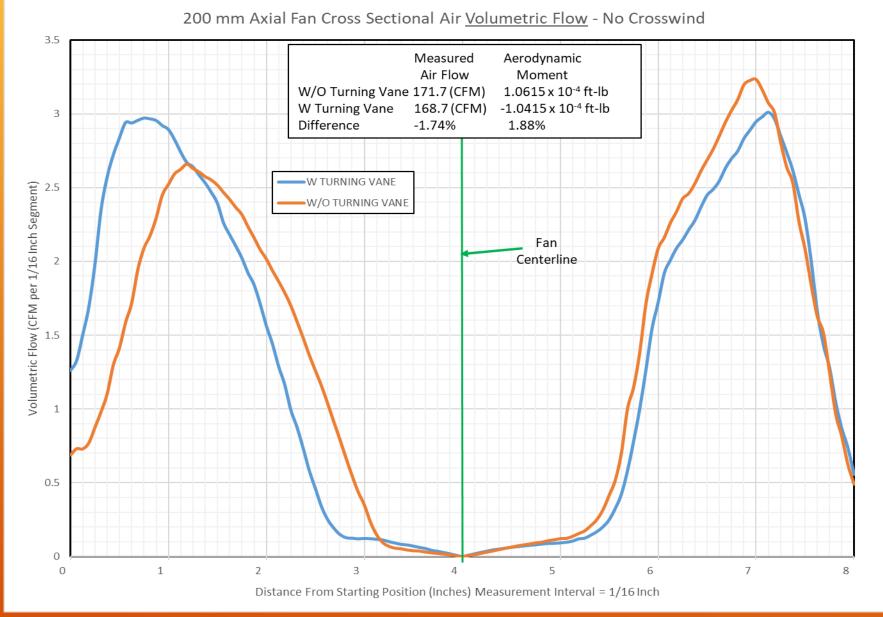
Revised number of measurement points per traverse = 129, interval = 1/16", traverse length = 8" Measurements per traverse = 6,450 (50 x 129).

## WIND TUNNEL MEASUREMENT RESULTS

10/10/2018



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#### DERIVATION OF AERODYNAMIC MOMENT

Aerodynamic Moment is the sum of the unbalanced airflow mass pushing against the fan blades and creating a net bending moment against the Fan Shaft.

$$M_A = \left( \sum_{i=0}^{1=8"} D_i A F_i \right) \left( \frac{\rho}{\omega} \right)$$

Where:

 $M_A$  = Aerodynamic Moment (Ft. Lbs.)

i = Incremental Element (1/16" Measurement Increment)

 $D_i$  = Distance from Element to Fan Centerline (Ft)

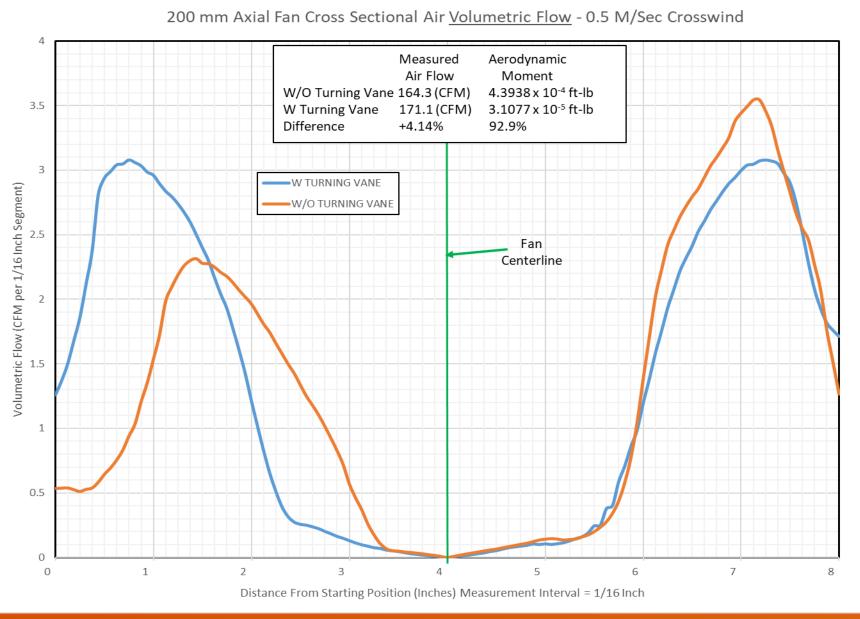
 $AF_i$  = Air Flow Through Element (Ft<sup>3</sup>/Minute)

 $\rho$  = Air Density (Lbs./Ft<sup>3</sup>)

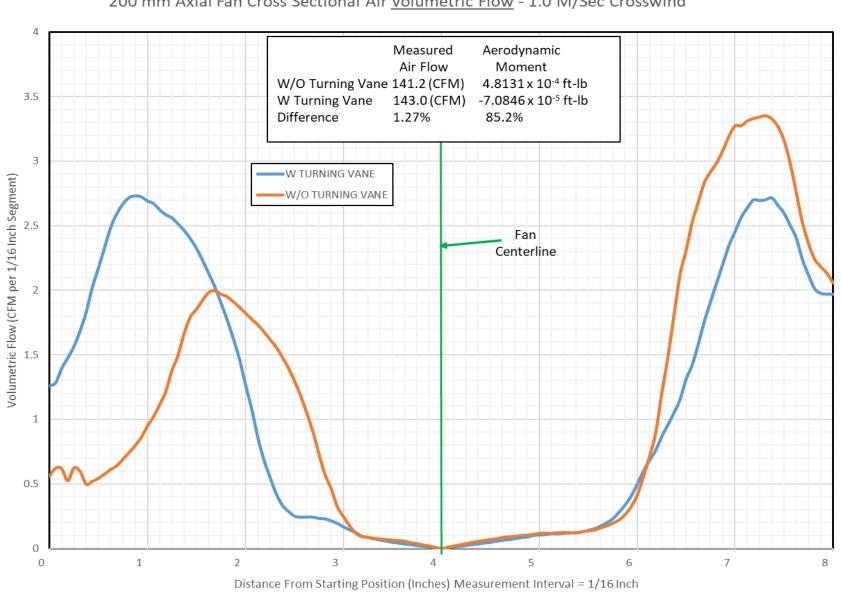
 $\omega$  = Fan Speed (RPM)

(Assumes an even bladed axial fan.)



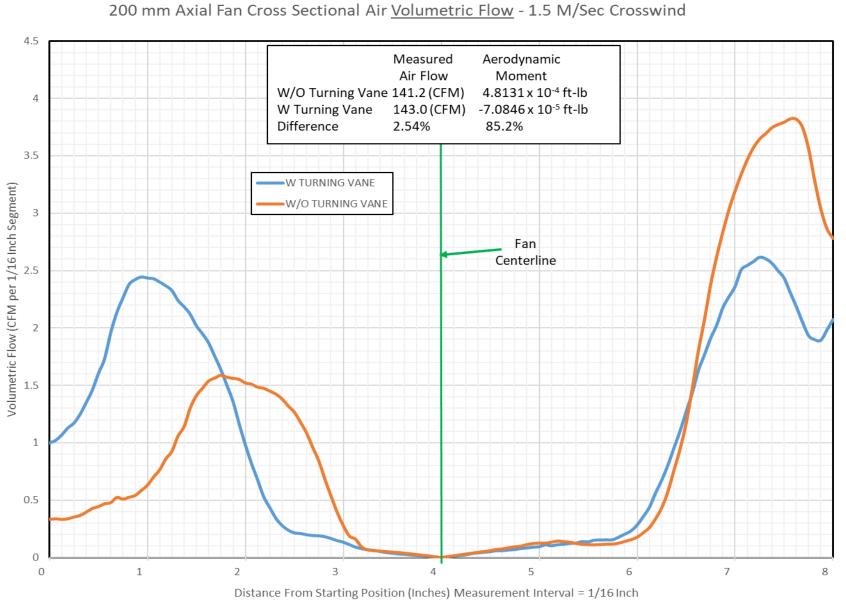


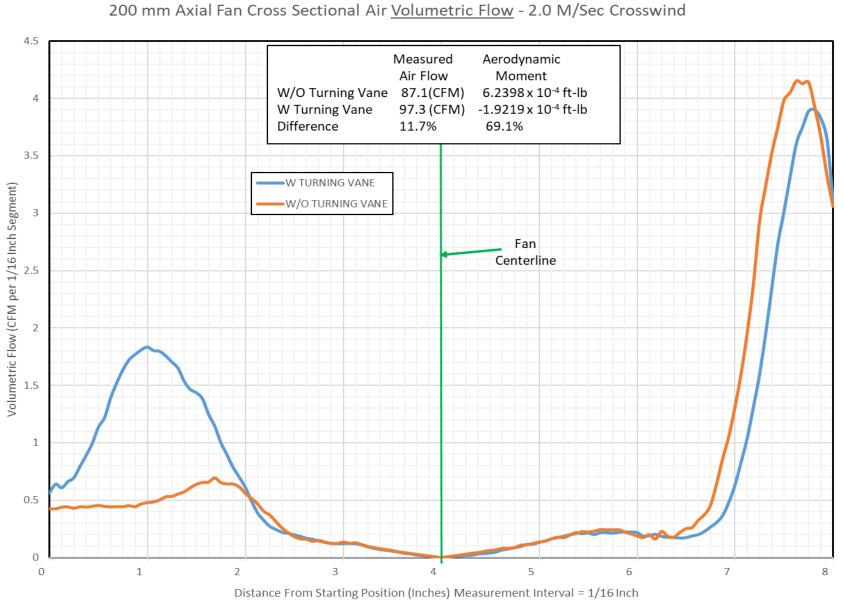
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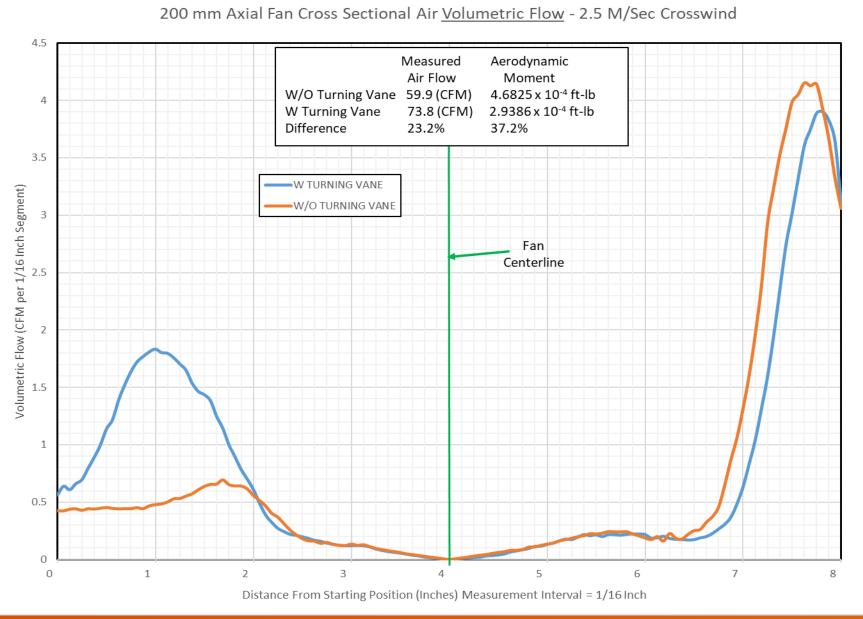


200 mm Axial Fan Cross Sectional Air Volumetric Flow - 1.0 M/Sec Crosswind

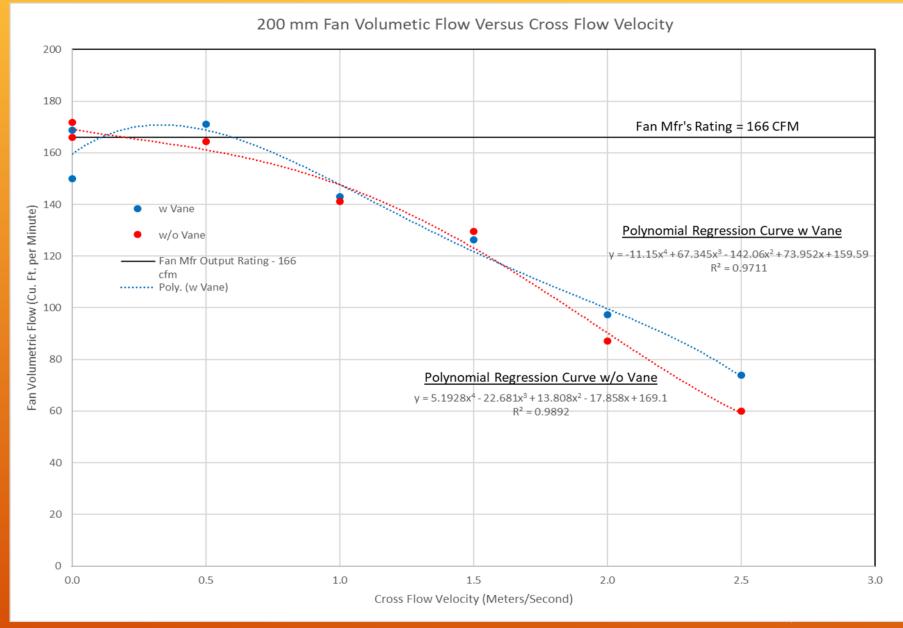
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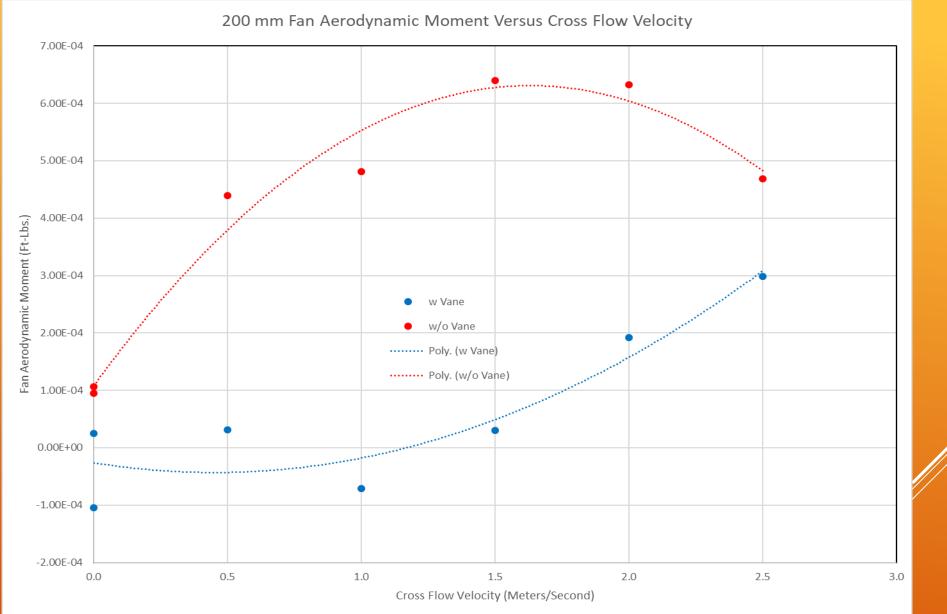




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## REVISED APPARATUS PROVIDES QUANTITATIVE ANSWERS

- How much does the Axial Wind Turning Vane obstruct air flow when there is no crosswind? Answer: Cross sectional physical area of the Turning Vane is about 5%, the measured airflow with the Turning Vane mounted is a reduction of 1.74%
- Is there a break-even point where the axial fan air flow actually improves with the use of the Axial Wind Turning Vane? Answer: Break-even for the Wind Tunnel test apparatus is below 0.5 M/Sec. Axial Fan output air flow with the Turning Vane attached tracks closely to air flow of axial fan without the Turning Vane.
- Can the Axial Fan aerodynamic imbalance be measured? **Answer: Yes** provided ambient air temperature and pressure measurements are taken.
- How much does the Axial Wind Turning Vane reduce the Axial Fan aerodynamic imbalance? Answer: It varies with cross wind velocity but the Axial Wind Turning Vane reduces aerodynamic imbalance by as much as 92.9%.





### THE AXIAL WIND TURNING VANE REDUCES AERODYNAMIC IMBALANCE BY AS MUCH AS 92.9%.

THE AXIAL FAN INLET WIND-TURNING VANE ASSEMBLY WILL DRAMATICALLY REDUCE YOUR ACC FAN WIND-RELATED VIBRATIONS.



### AXIAL WIND TURNING VANE INTELLECTUAL PROPERTY RIGHTS

- Axial Fan Inlet Wind-Turning Vane Assembly, US Patent No. 9,593,885 B2, International Patent Application No. PCT/US2014/053353 (WIPO WO2015031723), granted 2017-03-14.
- Chinese Patent Application Serial No. 201480059969.4 Axial Fan Inlet
  Wind-Turning Vane Assembly, Notification to Grant Patent Rights, July 9, 2018
- European Patent Office Intention to Grant a European Patent Axial Fan Inlet Wind-Turning Vane Assembly – Application No. 14 839 580.9 – 1008, granted 20 Aug 2018

Intellectual Property Rights Inquires – Contact: M. Cuerdon, 720-357-3786, Mcuerdon@aol.com

