



WELCOME TO *Galebreaker*[®]

Wind Effects on an ACC at a Geothermal Power Station

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Agenda

- Geothermal Power basics
- Geothermal power in the US
- Videos – smoke test, dry ice
- CFD study of wind effect at a small geothermal power plant
- Mitigation strategies

Geothermal Basics

- Three types of geothermal power plants
- Dry steam- steam direct from an underground reservoir to turn the generator turbines
- Flash Steam- high pressure hot water from very deep wells flashed into steam to drive the turbines. Steam condenses and water injected back into the ground for another cycle.
- Binary-cycle- Geothermal hot water used to convert another liquid like isobutane (via a heat exchanger) into a gas (steam) that drives the turbine. The water is reinjected back into the well.

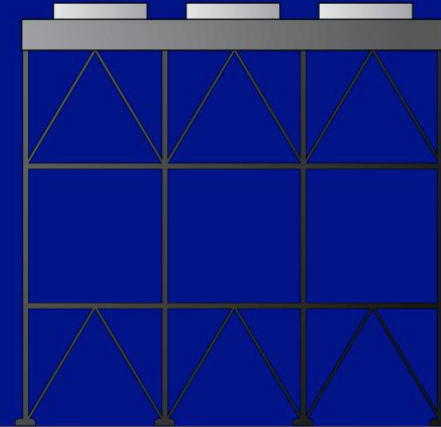
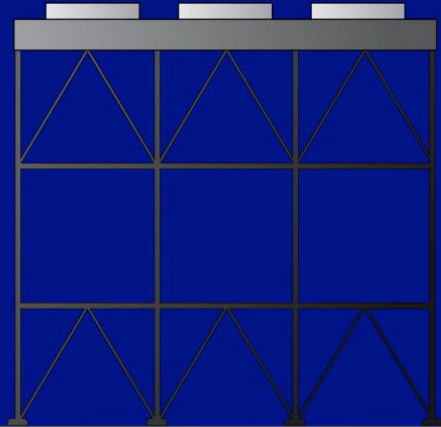
Geothermal in the US

- 3.9 GW as of 2023
- Only 4%
- The Geysers, commissioned in 1960, probably still the largest geothermal field in the world, 1500MW. 22 units
- Geothermal; capacity factor of 65%. Much higher than solar or wind.
- Big Beautifull Bill- controversial, extends tax credits for geothermal, others expire.
- Huge growth potential, new technologies in drilling, monitoring well production.

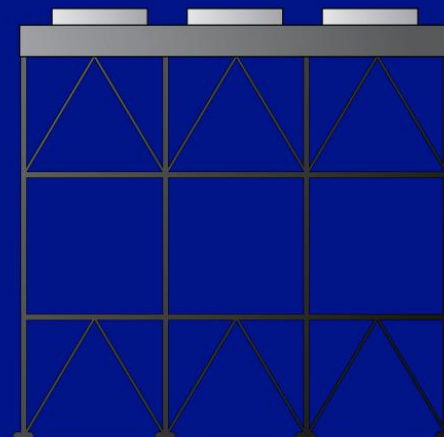
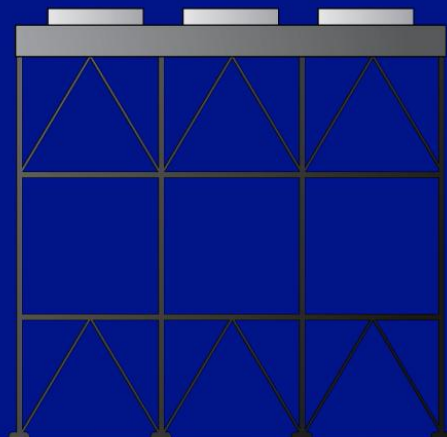
Geothermal in the US

- For Galebreaker, similar wind effects to what we see on our LNG projects
- Large ACHE close to each other; reduce expensive pipework
- Geothermal- typically in desert locations with very high seasonal prevailing winds.
- Growing market

HOT AIR RECIRCULATION



HOT AIR MIGRATION



Wind Tunnel and Smoke Testing in the Field

Video of Wind Tunnel Testing at the U.C. Davis ABL Wind Tunnel Showing Smoke Testing At a 9 m/s Wind Speed with and without wind screens under the ACC (video provided upon request)

**Video of Field Testing Using Dry Ice to Demonstrate Recirculation
between Modules On An ACHE (Video Provided Upon Request)**

**Video of Smoke Testing on an Operating ACC with and without Windscreens
(Video Provided Upon Request)**

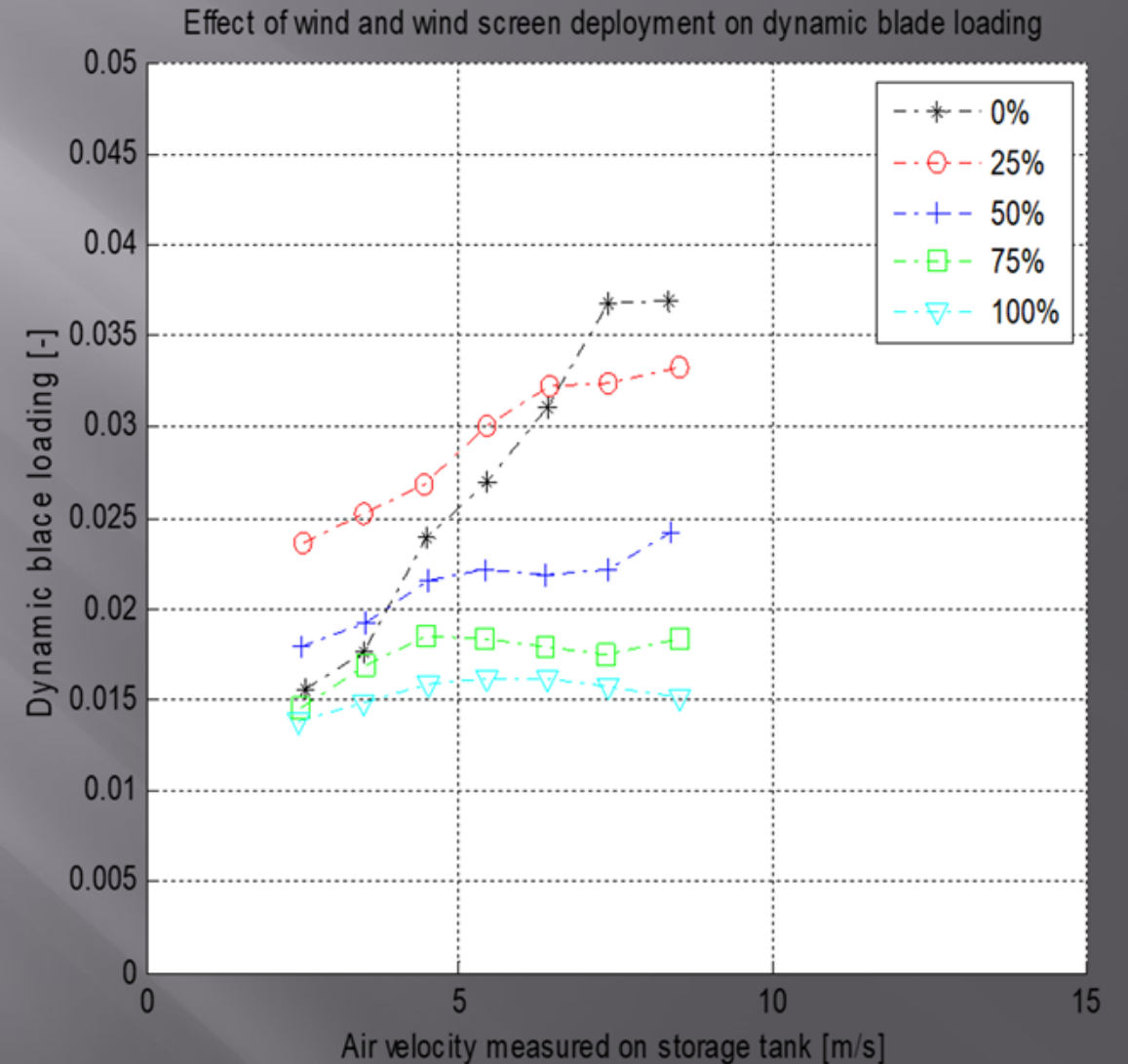
Dynamic Blade Loading Fan 3.4

Loading Increases at
Higher Wind Speeds

Significant Influence of
Wind Screen

Screen Deployment
Reduces Loading

Fully Deployed Dynamic
Blade Load Reduced by a
Factor 2-3

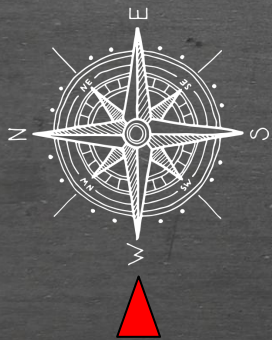


An existing solution for
mitigating hot-air
recirculation



Analysis Via CFD



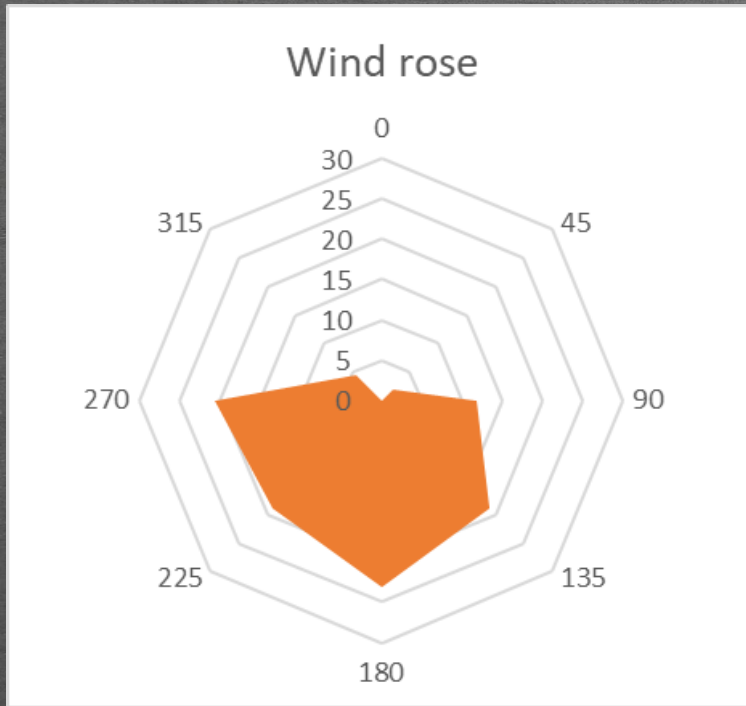


Wind direction

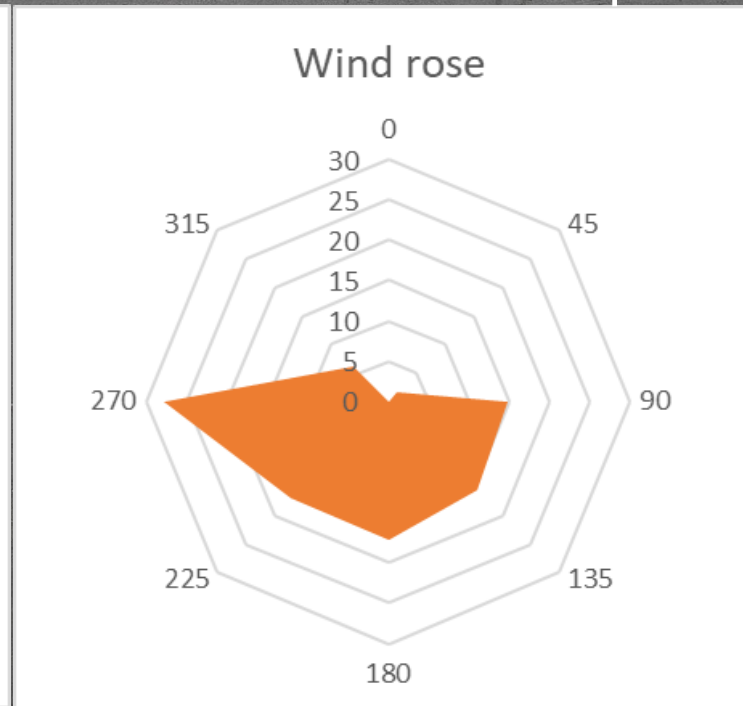


Site Wind Rose

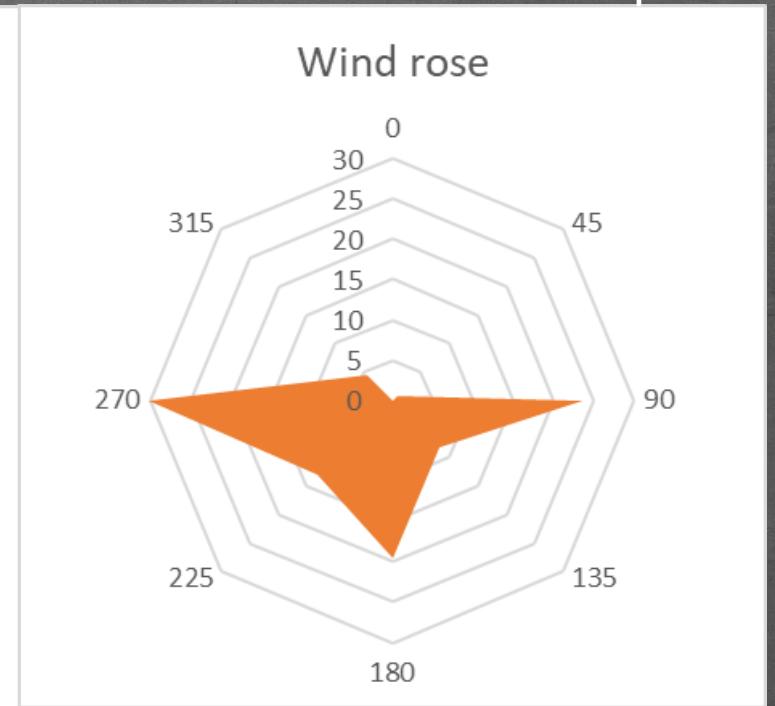
All events



Filtered for WS > 5 mph



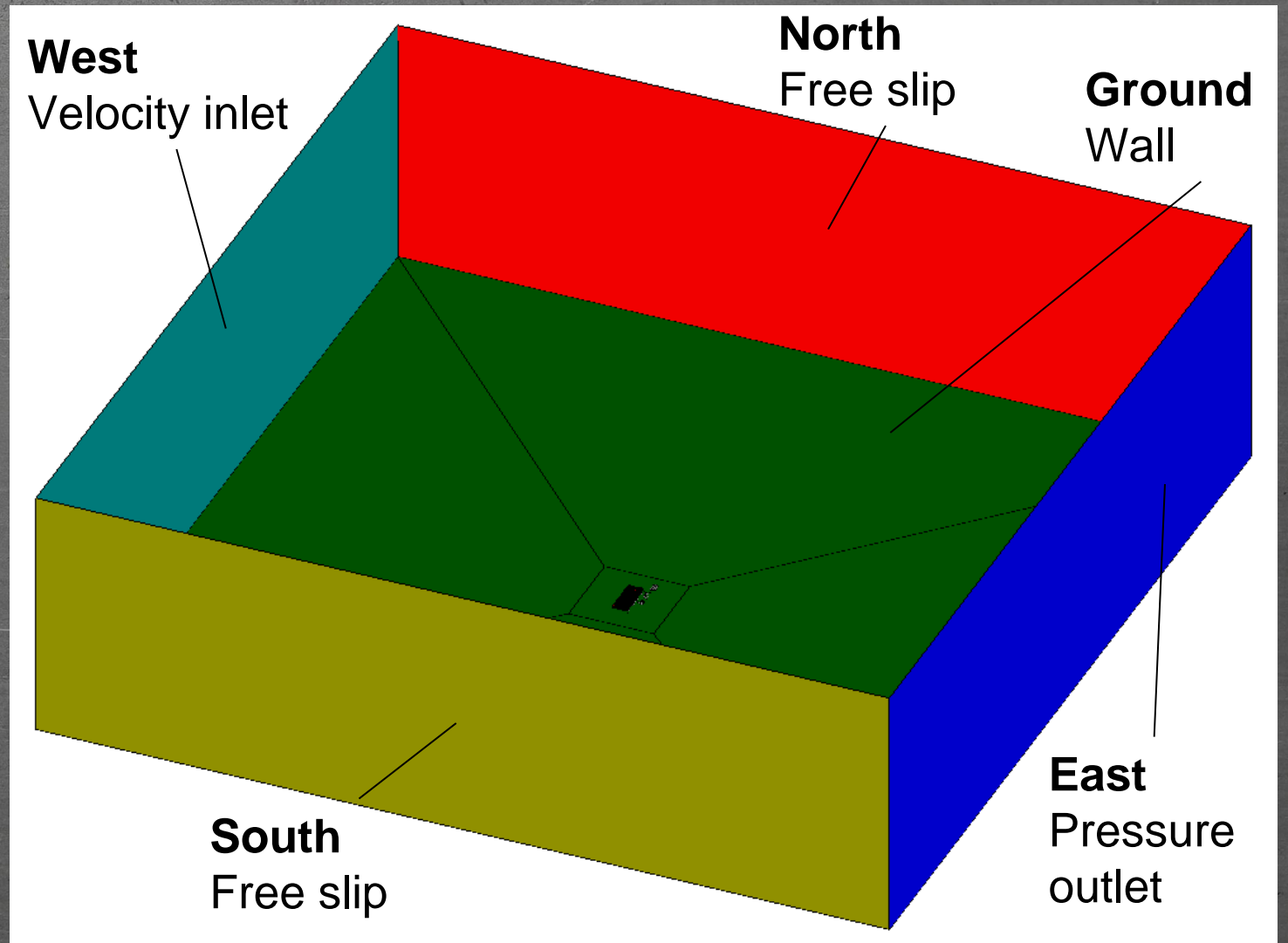
Filtered for WS > 14 mph



Modelling details

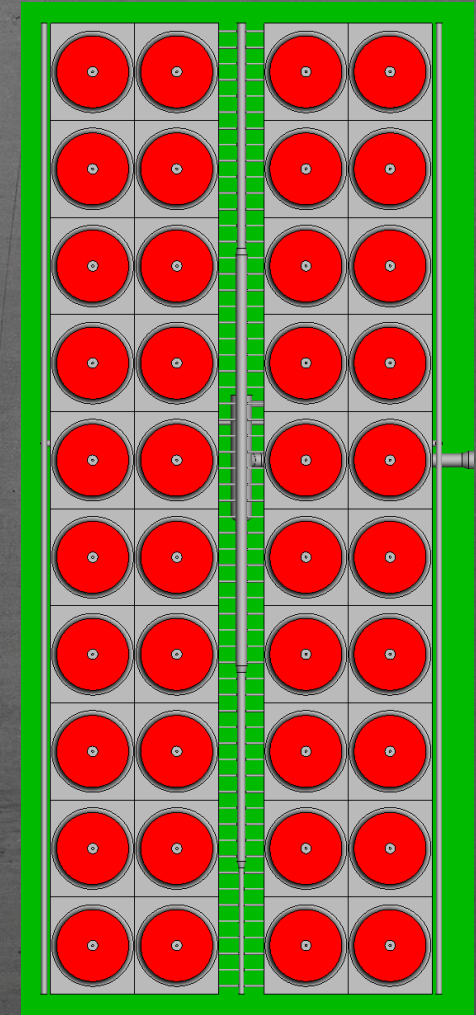
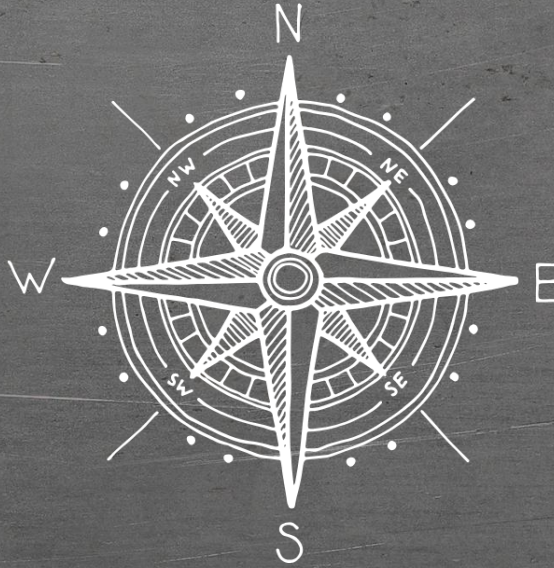
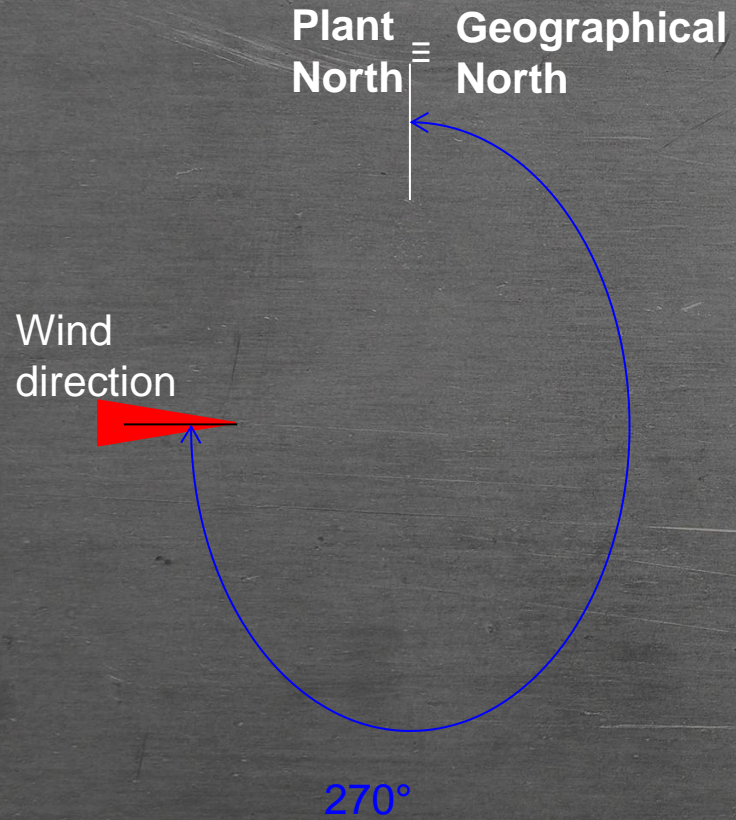
Domain
definition

2km x 2km x
600m height



Modelling details

ACC geometry



Modelling details

ACC geometry

Steam
pipes

Wall

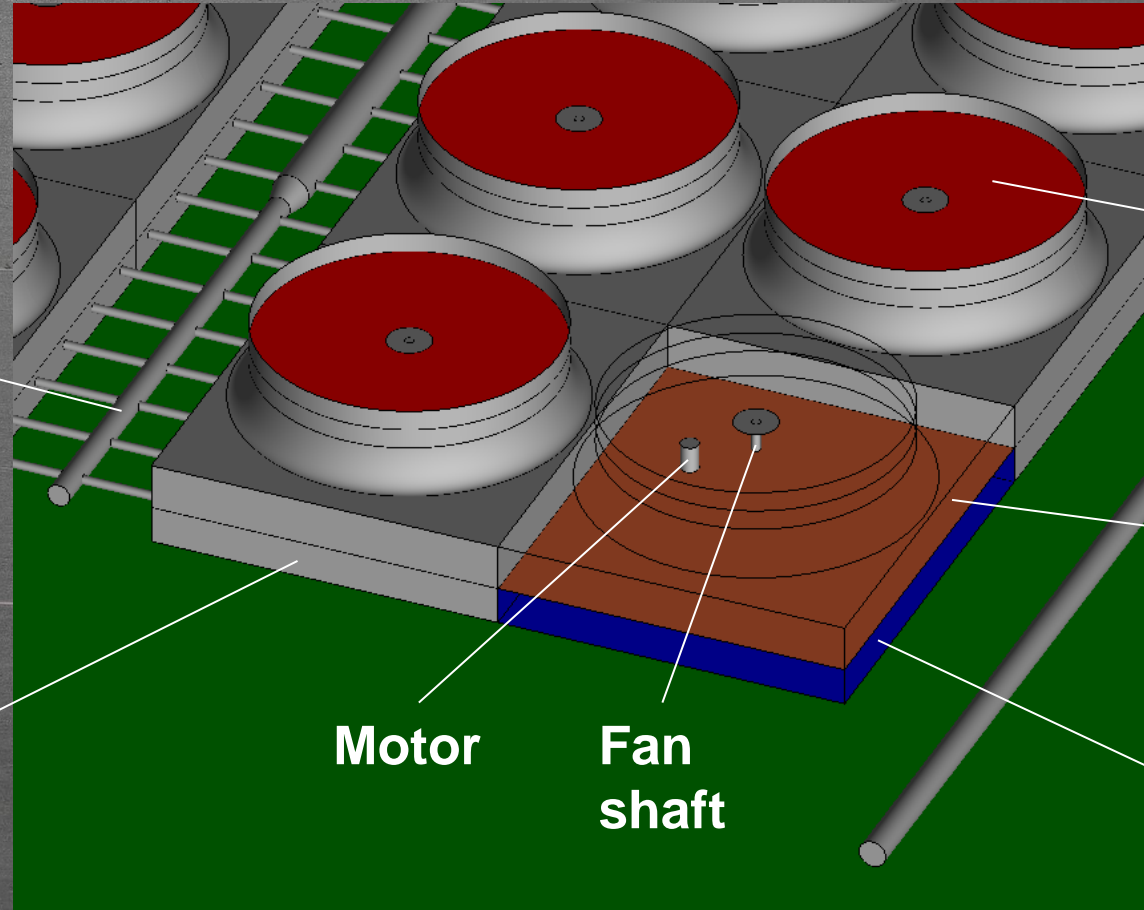
Motor

Fan
shaft

Fan

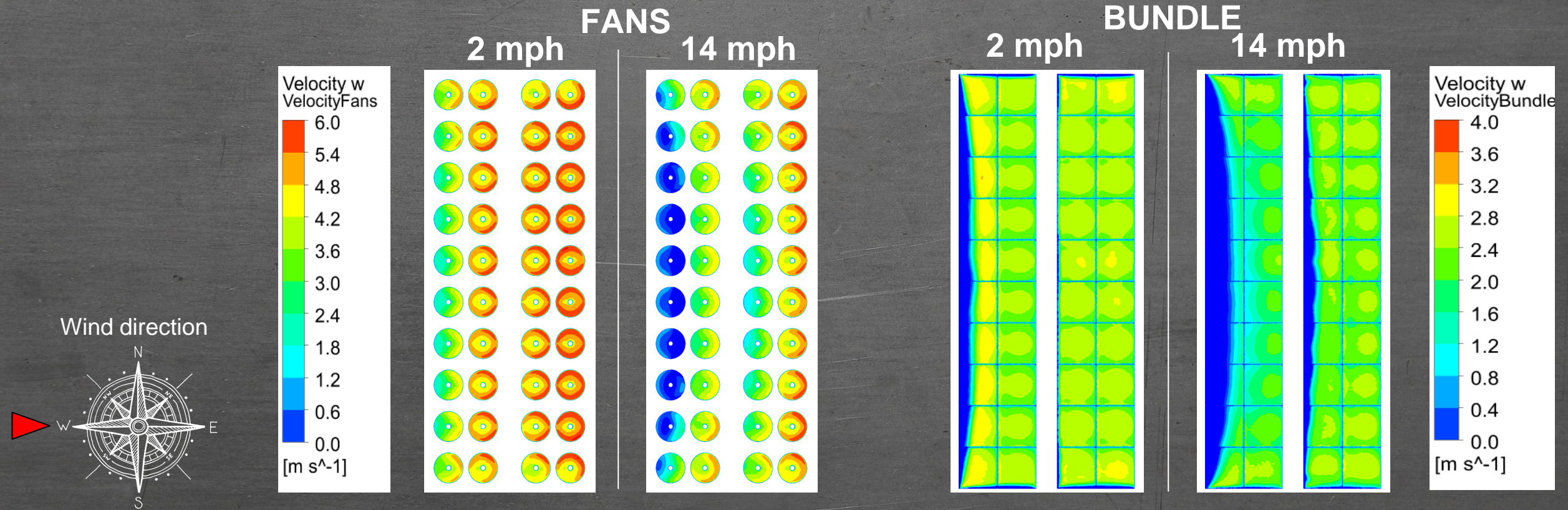
Bundle
out

Bundle
in



Baseline results

Vertical velocity on fans and bundle inlet - The misbalance of airflow generates an overall reduction of the flow



Baseline results

Temperature on fans and bundle as a result of recirculation and backflow



Baseline results

Mass flow rate

At both wind speeds the baseline configuration (L00) shows a deficit if compared to design condition

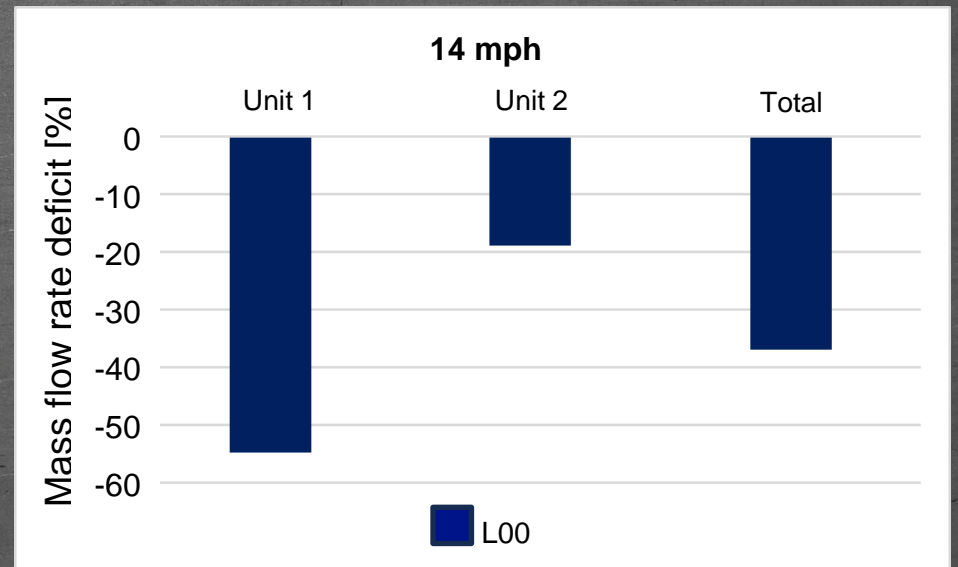
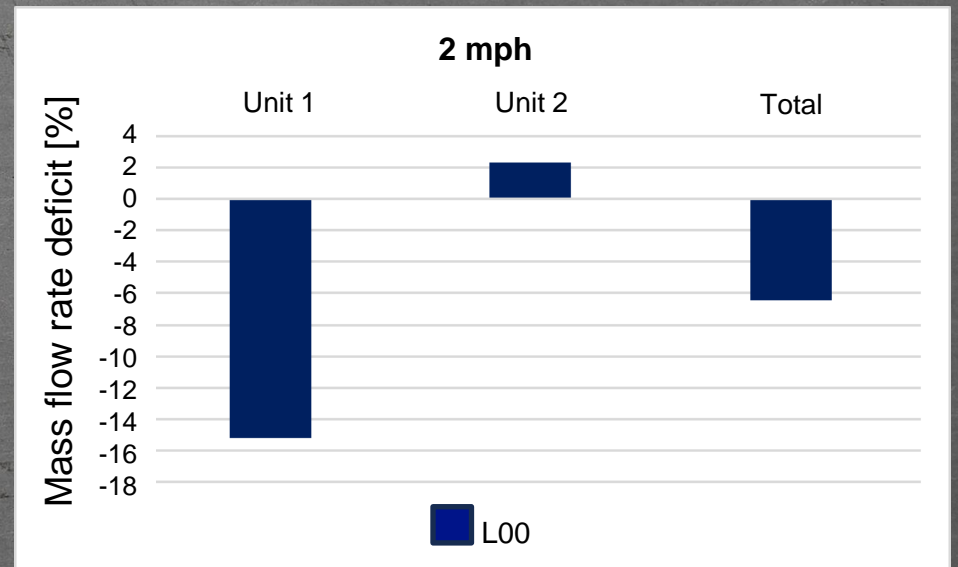
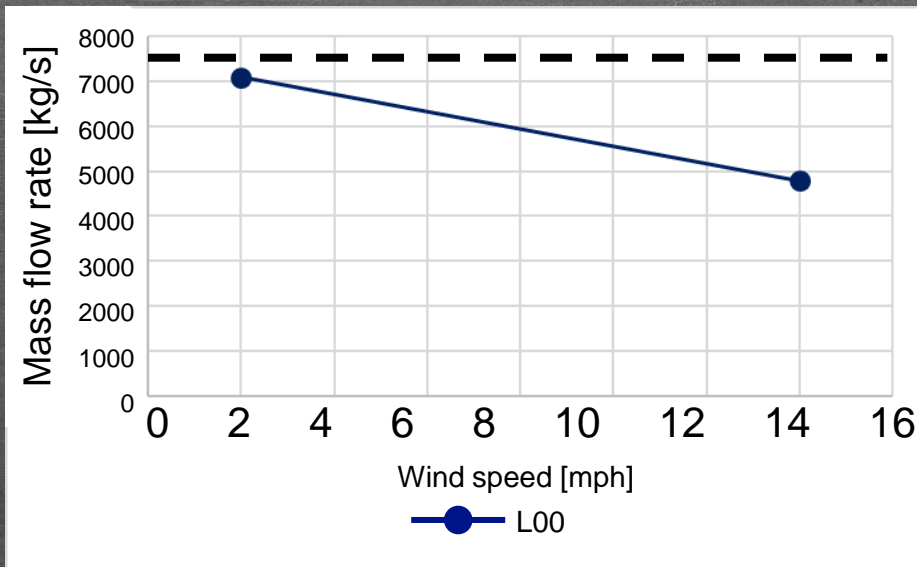
@2 mph the deficit is **-6.44%**

@14 mph the deficit is **-36.87%**

$$m_{\text{deficit}} = \frac{m - m_{\text{design}}}{m_{\text{design}}}$$

Downstream unit is delivering more than design air flow

Design flow rate =
7570 kg/s



Baseline results - L00

Recirculation

Recirculation is a measure of the quantity of spent air ingested at the fan

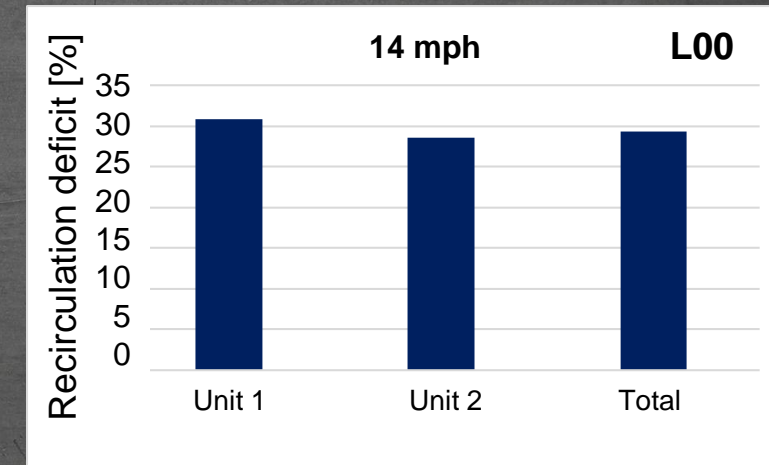
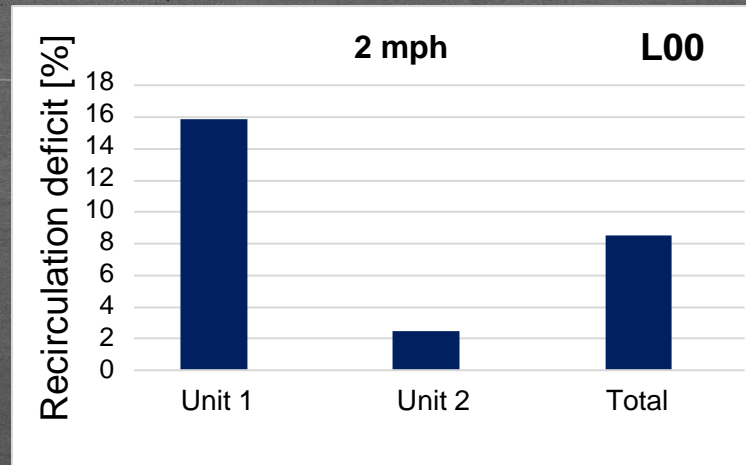
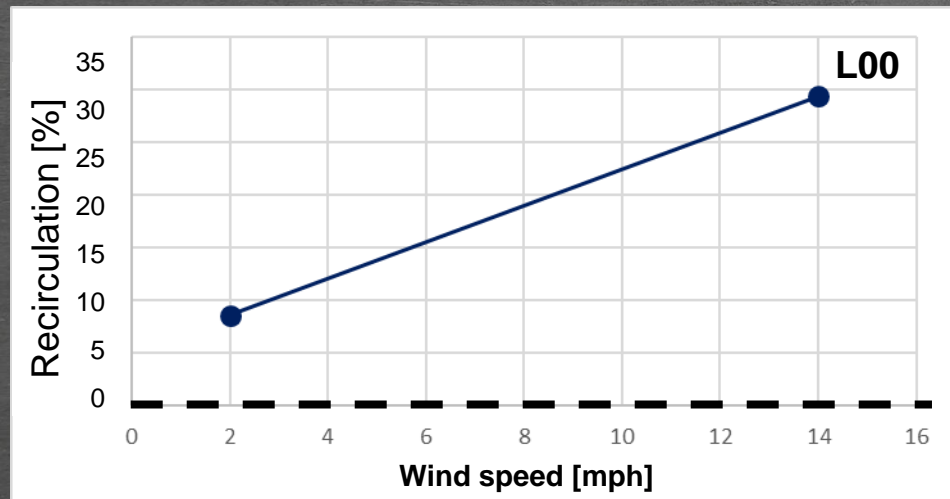
It is defined based on the temperature level at the fan

$$R = 100 \cdot \frac{T_{fan} - T_{amb}}{T_{bundle} - T_{amb}}$$

L00 shows a considerable recirculation at both wind speeds:

@2 mph the deficit is **8.55%**

@14 mph the deficit is **29.36%**



Baseline results - L00

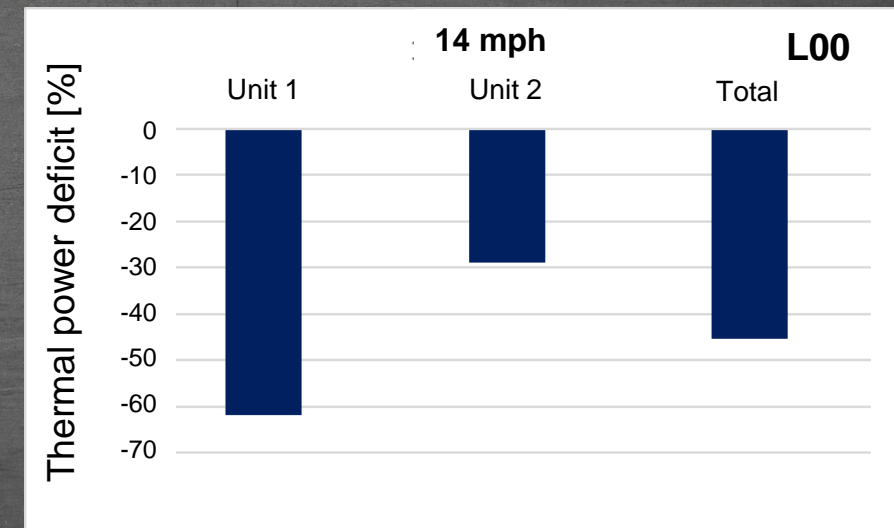
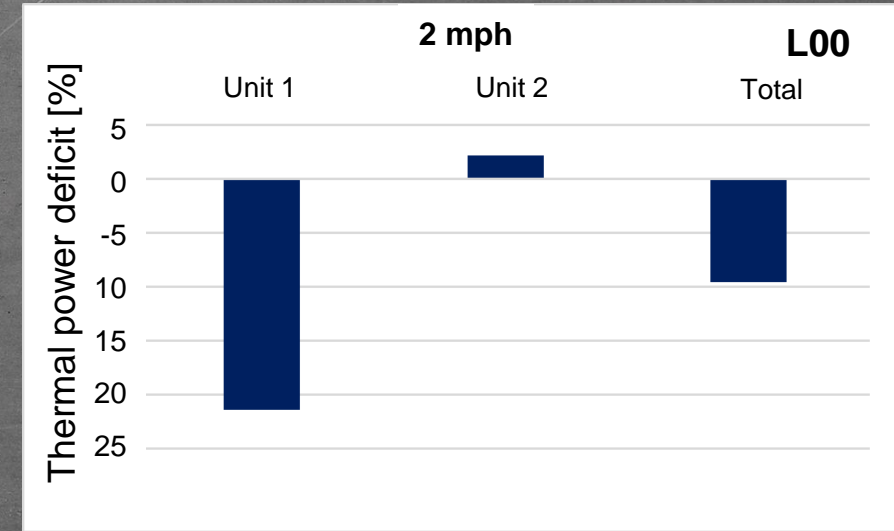
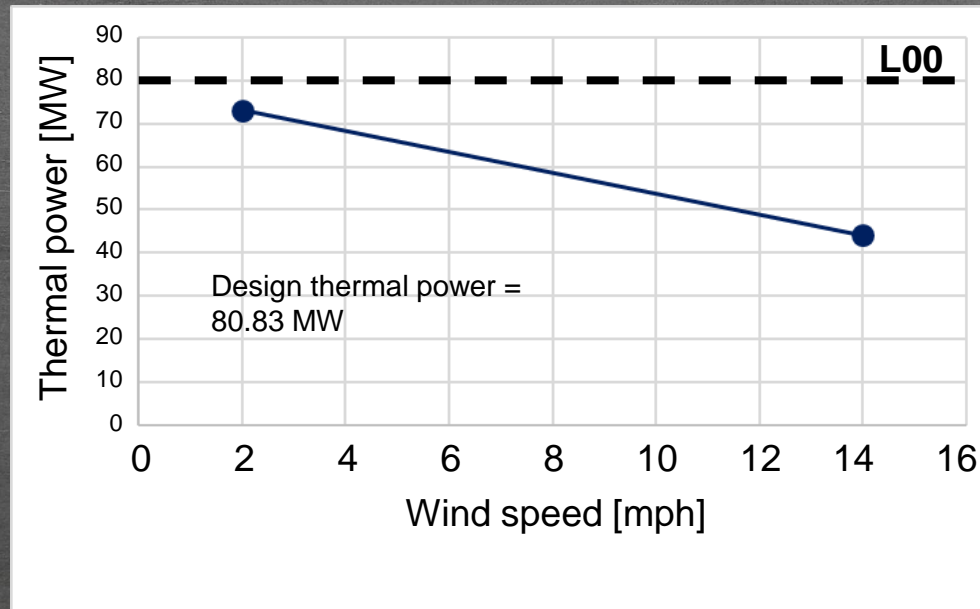
Thermal power

At both wind speeds L00 configuration shows a deficit if compared to design condition

@2 mph the deficit is **-9.59%**

@14 mph the deficit is **-45.37%**

$$\dot{Q}_{deficit} = \frac{\dot{Q} - \dot{Q}_{design}}{\dot{Q}_{design}}$$

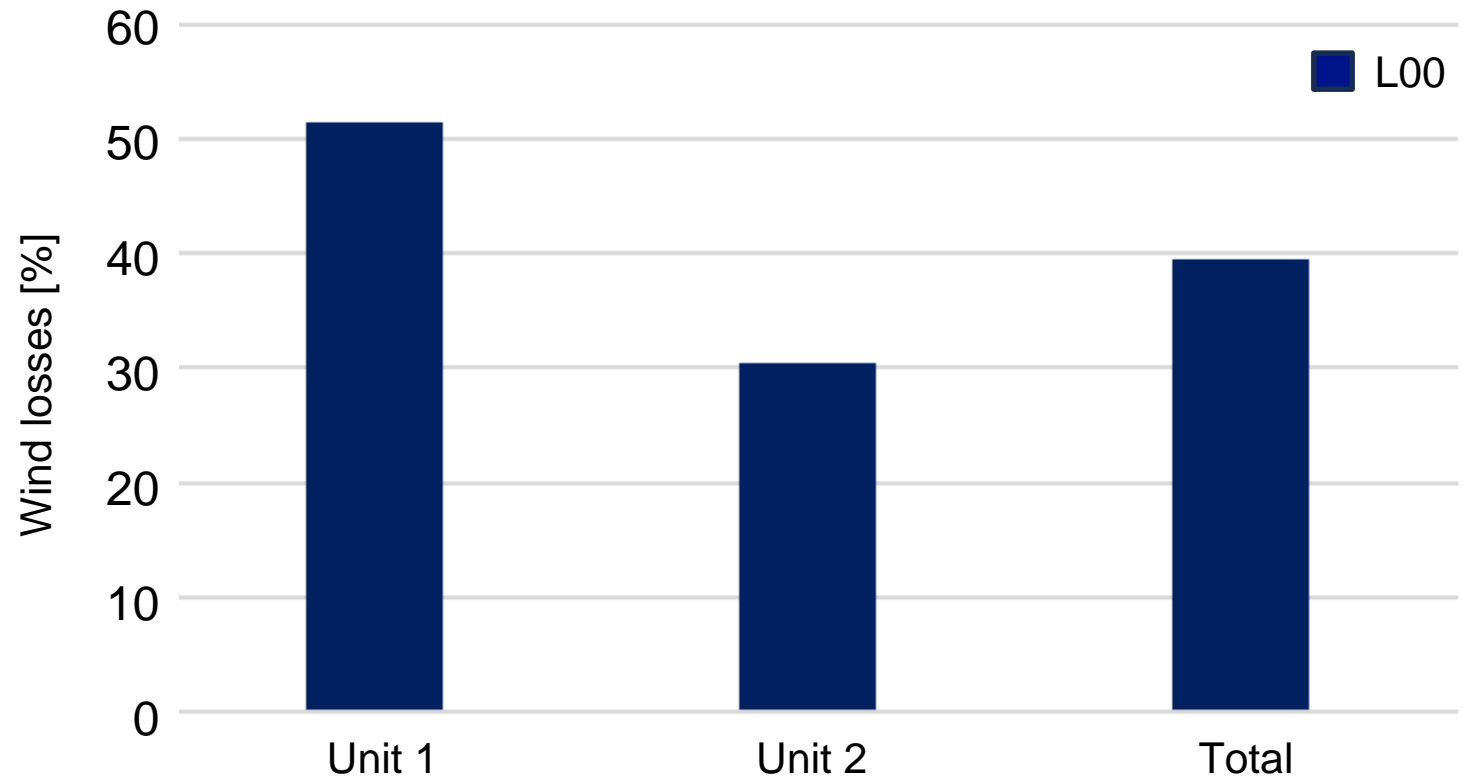


Baseline results - L00

Wind loss

@14 mph the wind loss for the L00 configuration is **39.57%**

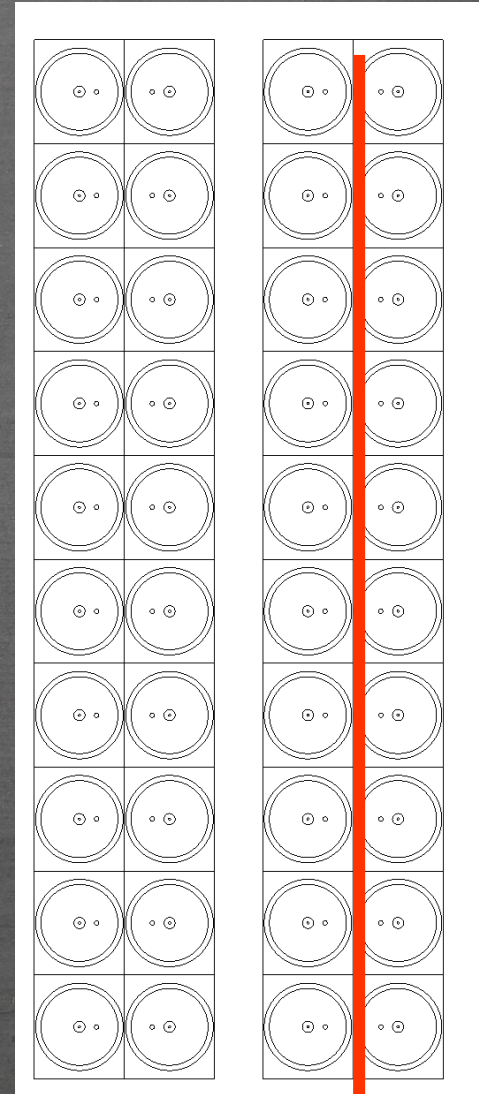
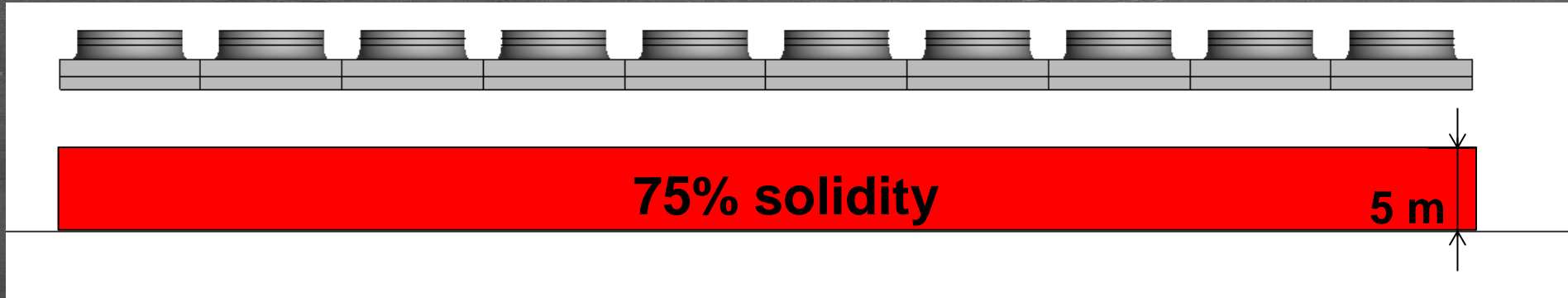
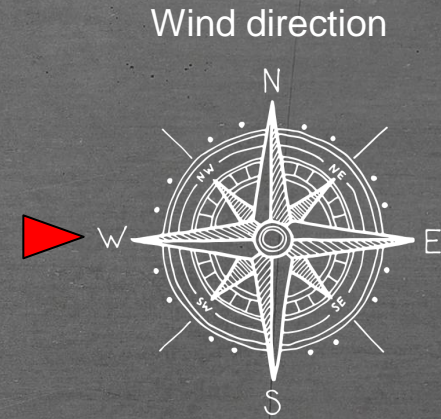
$$\text{Wind losses} = \frac{\dot{Q}_{2\text{mph}} - \dot{Q}_{14\text{mph}}}{\dot{Q}_{2\text{mph}}}$$



Proposed windscreens configurations

To increase the ACC performance the following windscreens configuration are proposed:

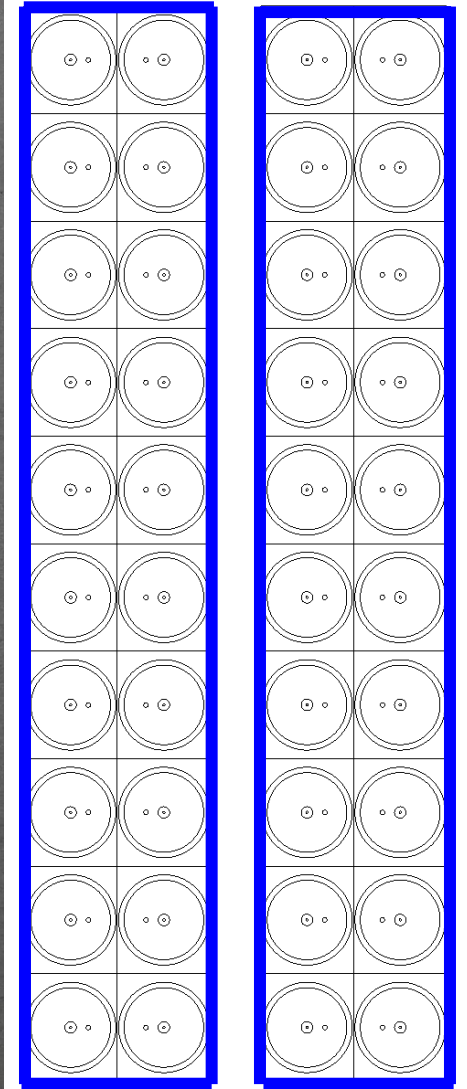
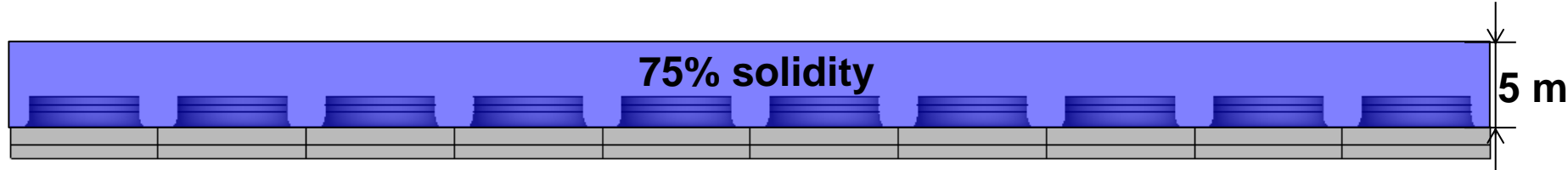
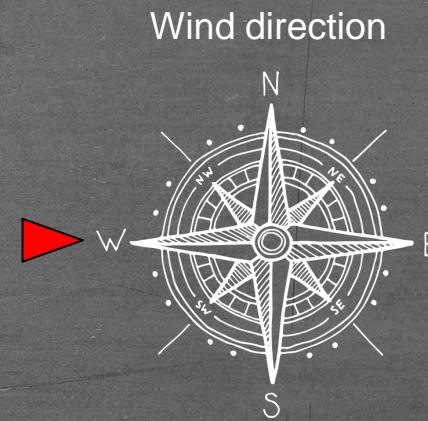
- L01 Wind wall



Proposed windscreens configurations

To increase the ACC performance the following windscreens configuration are proposed:

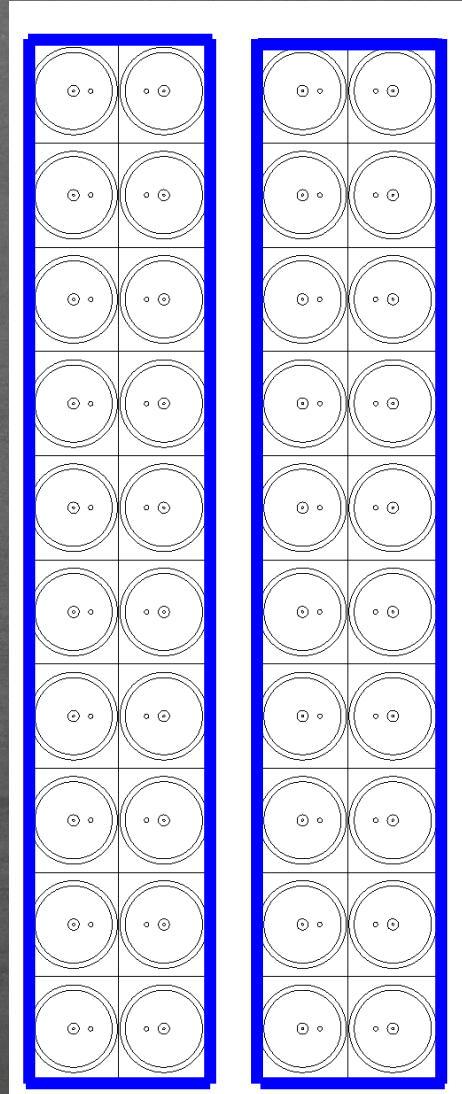
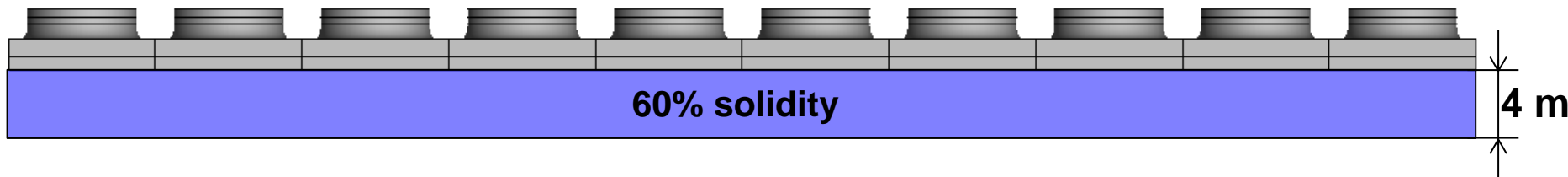
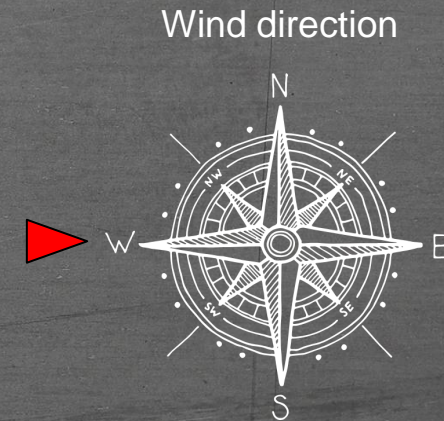
- L02 Chimney



Proposed windscreens configurations

To increase the ACC performance the following windscreens configuration are proposed:

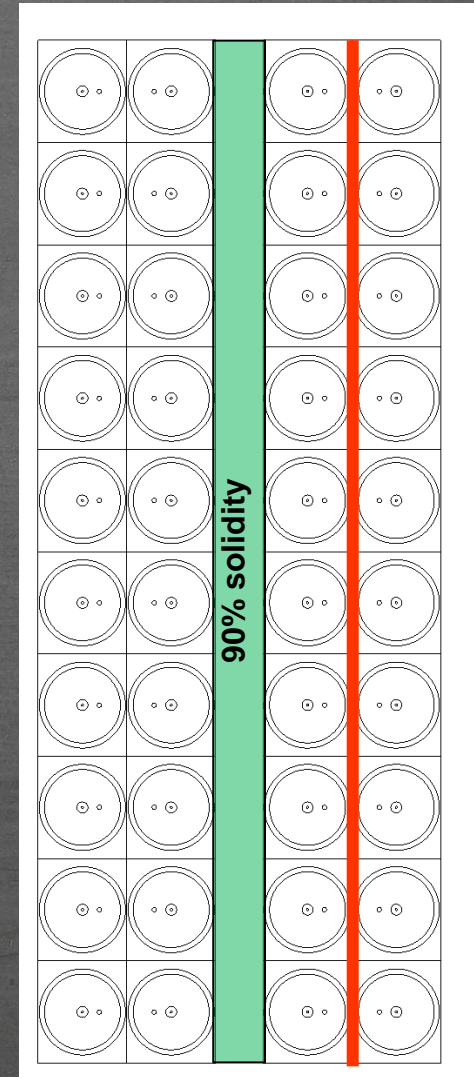
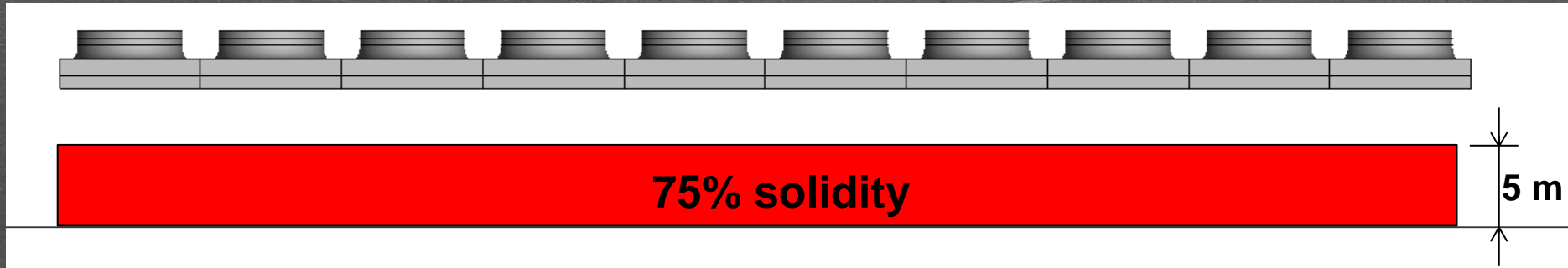
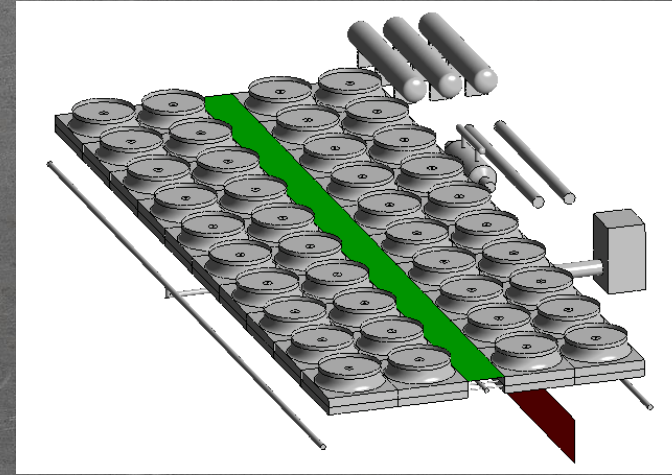
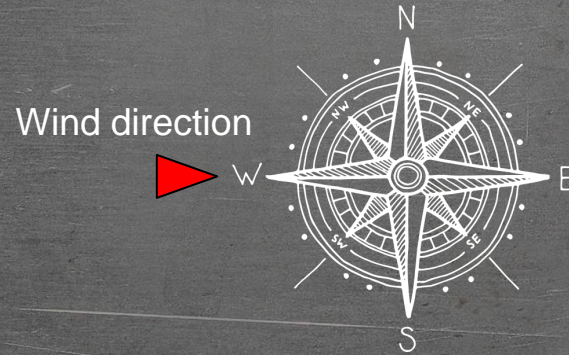
- **L03 Perimeter**



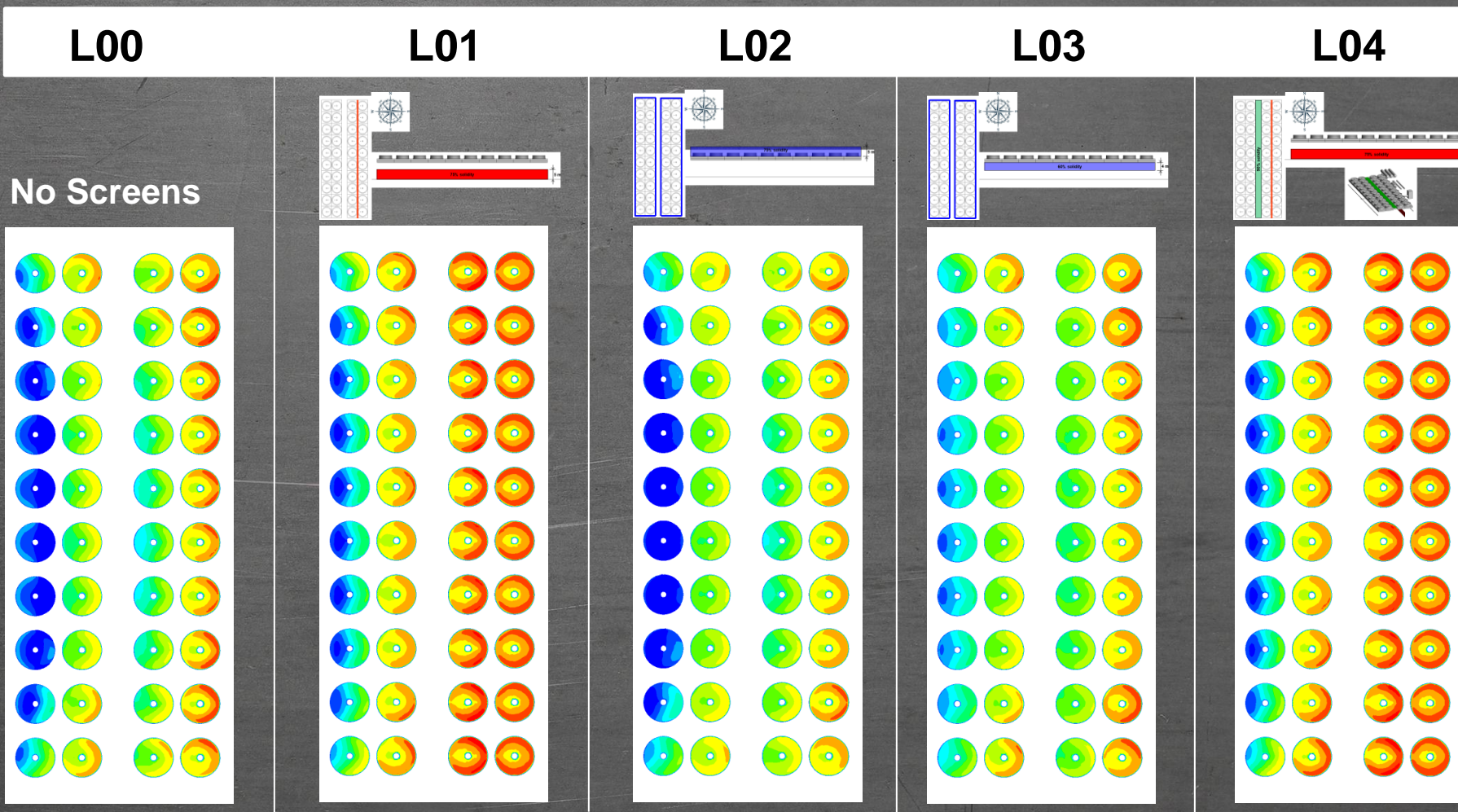
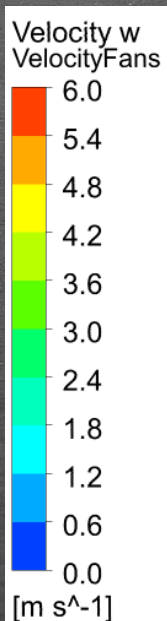
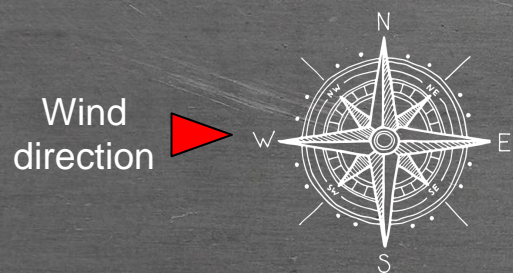
Proposed windscreens configurations

To increase the ACC performance the following windscreens configuration are proposed:

- L04 Wind wall and gap closure



Results – High wind speed



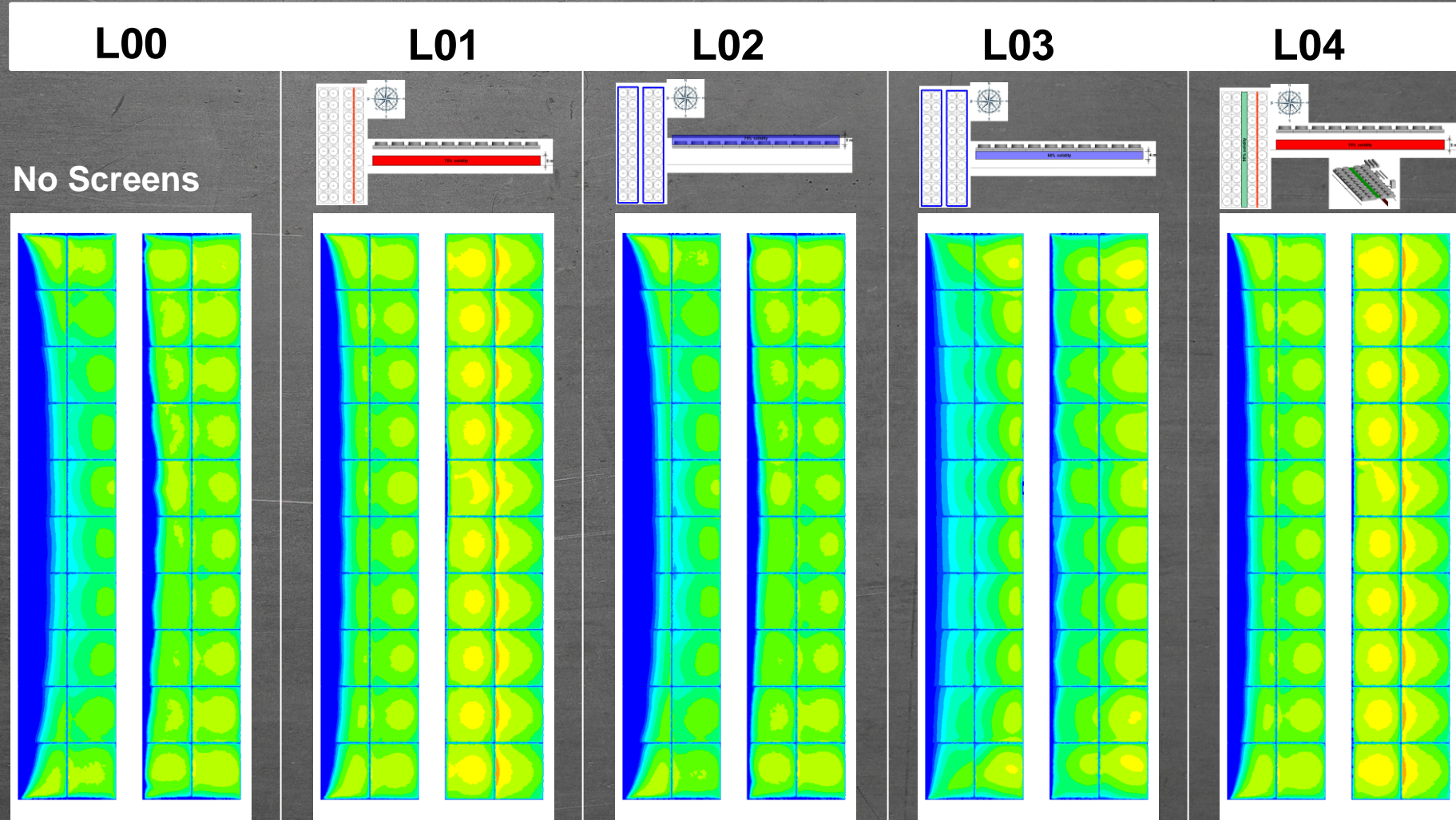
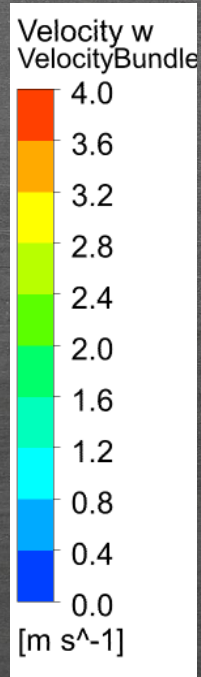
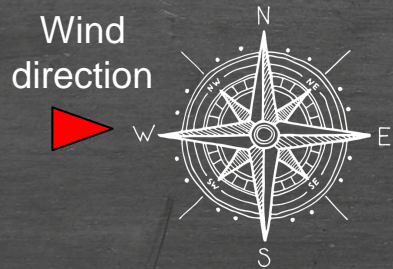
Results – High wind speed

Vertical velocity on bundle inlet

Red areas indicate high velocity (i.e. high flow rate)

Blue areas indicate low air velocity (i.e. low airflow or reversed flow)

The misbalance of airflow generates an overall reduction of the flow and creates fan imbalance



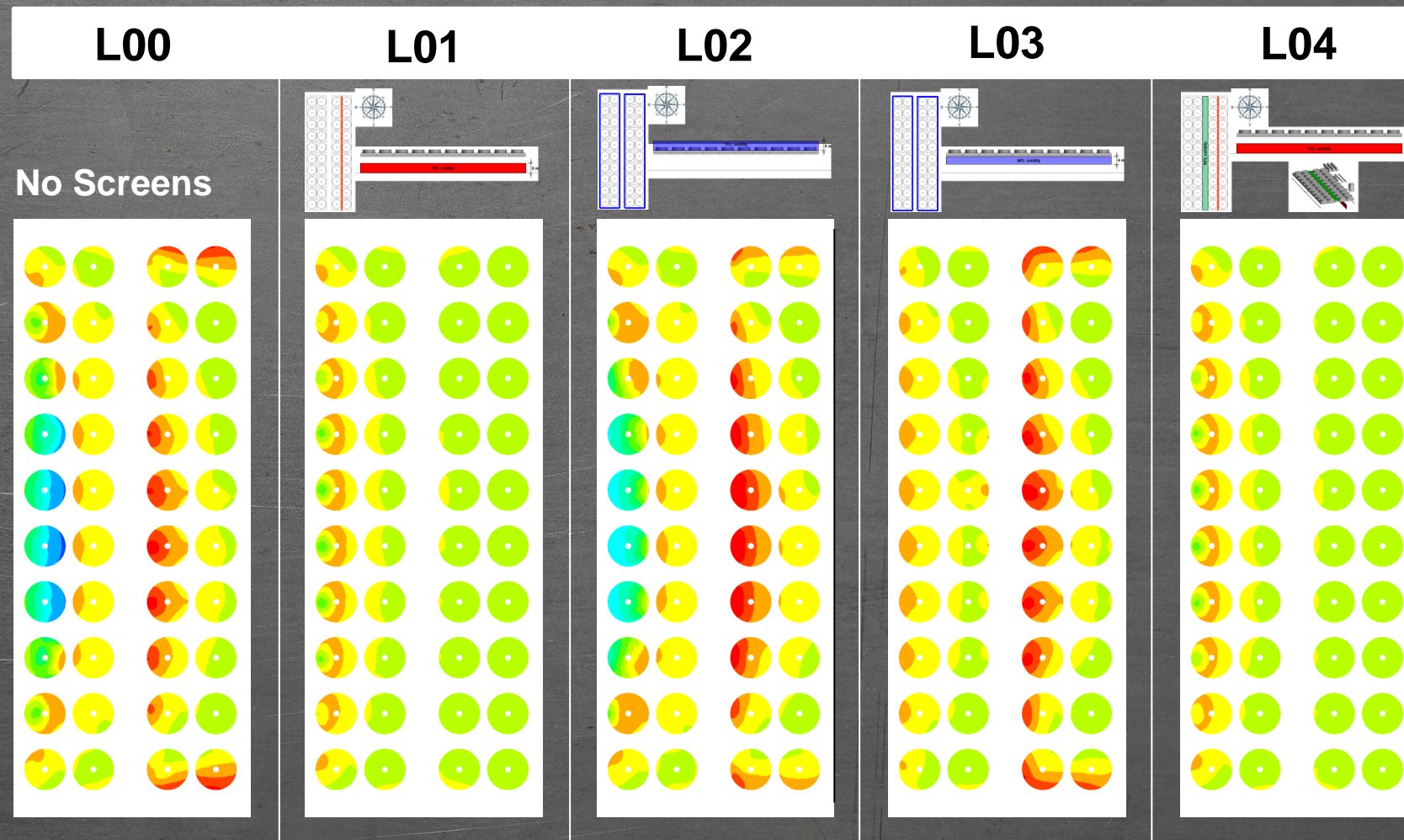
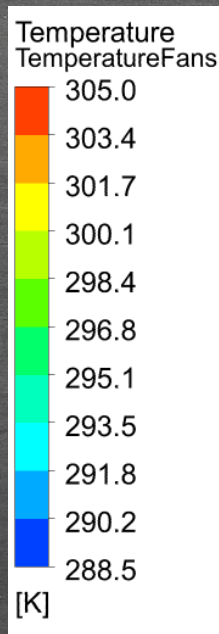
Results – High wind speed

Temperature on fans
as a result of
recirculation and
backflow

Design temperature is
299.16 K

**Low temperature
means backflow on
fans**

High temperature means
low air flow rate (higher
temperature increase)

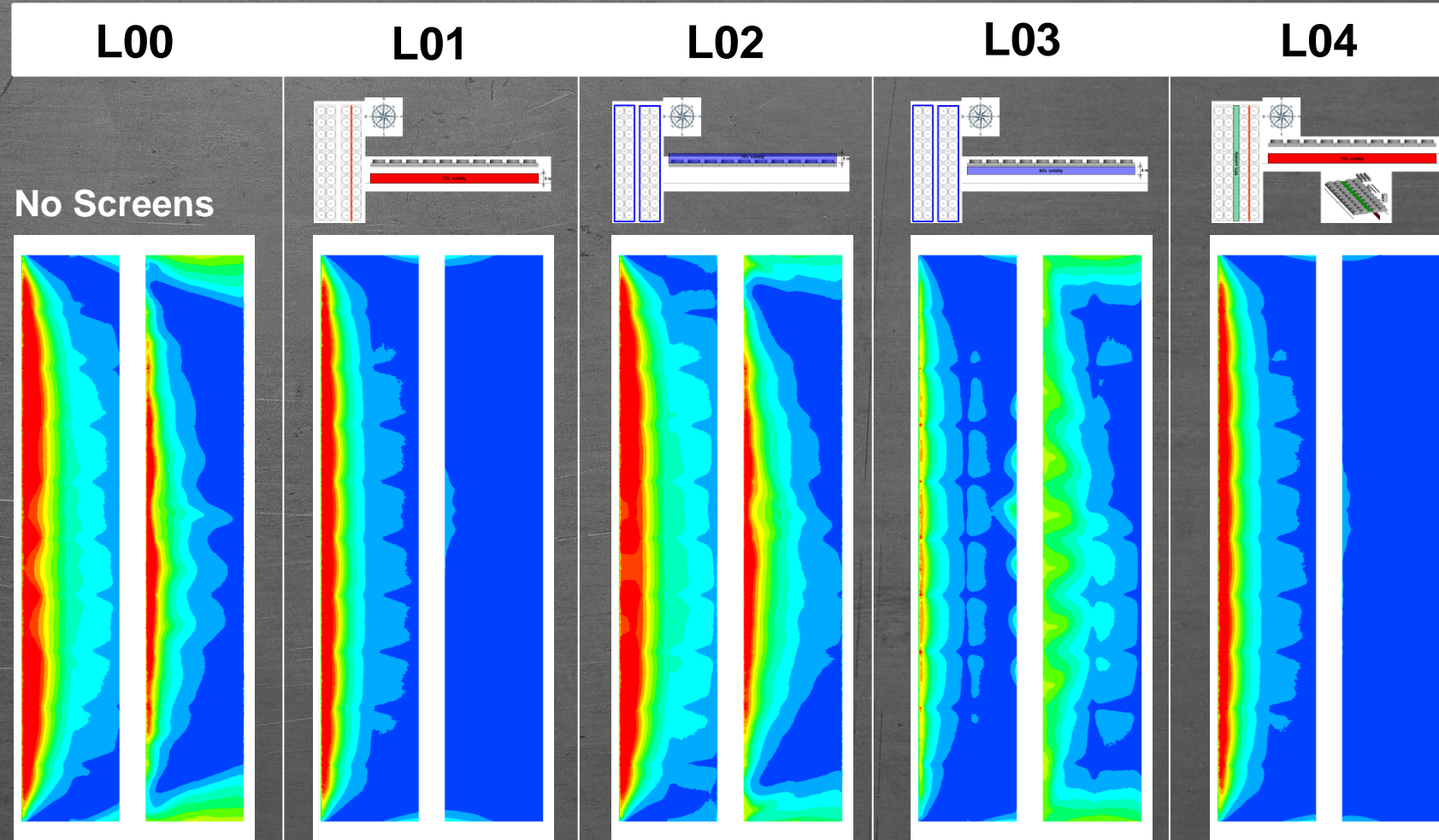
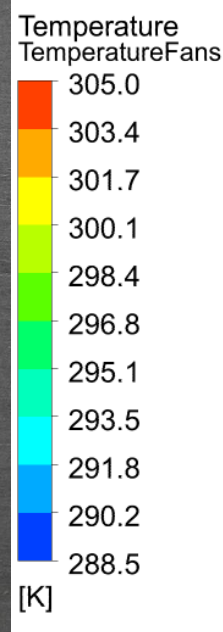
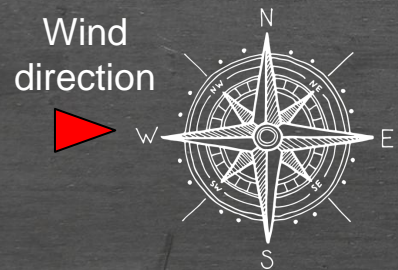


Results – High wind speed

Temperature on bundles inlet as a result of recirculation and backflow

Ambient temperature is 288.55 K

Any temperature higher than this reflect either a recirculation of spent air from the exterior or a backflow



Baseline results

Mass flow rate

At low wind speed, wind losses are minimal

Nevertheless, L01 (and L04) generates an additional benefits resulting in 3.69% gain

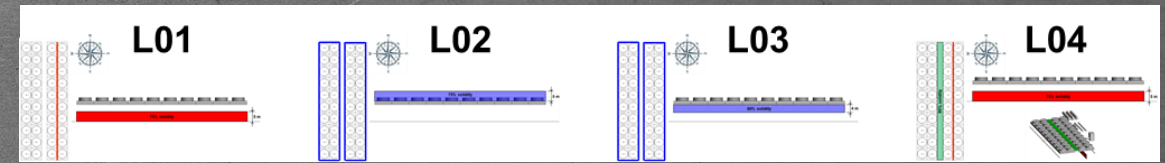
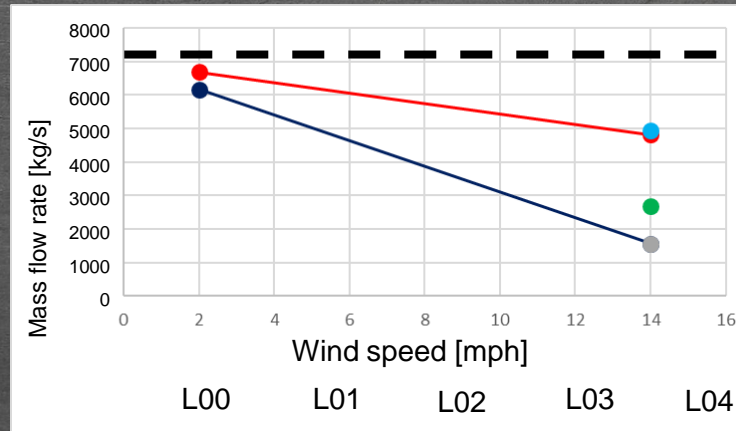
At high wind speed, wind wall based screens outperform the other layouts

The gain is high on both units with higher benefits on unit 1

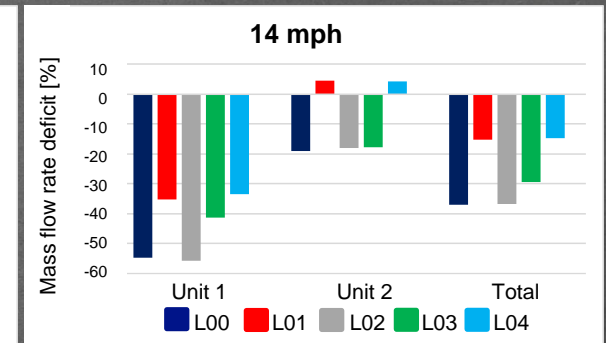
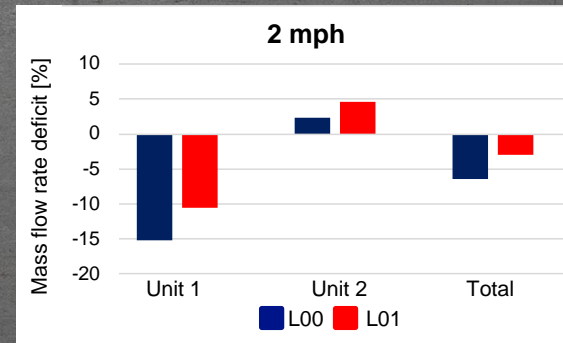
L04 shows the best results: @14 mph the gain is 35.27%

L03 is also showing some improvements

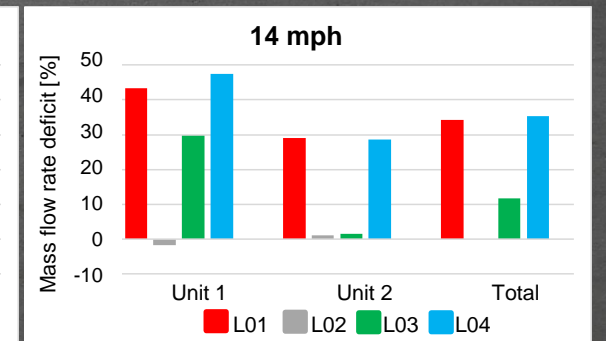
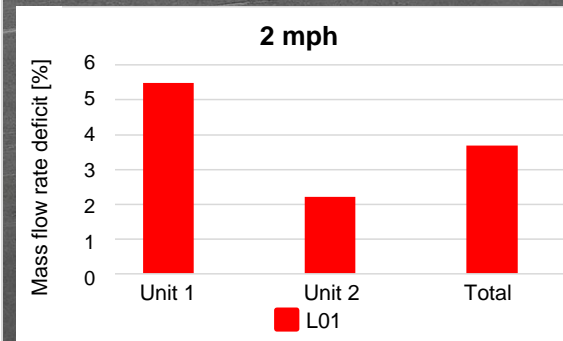
Design flow rate
= 7570 kg/s



$$m_{deficit} = \frac{m - m_{design}}{m_{design}}$$



$$m_{gain} = \frac{m_{L0x} - m_{L00}}{m_{L00}}$$



Baseline results

Recirculation

Recirculation is a measure of the quantity of spent air ingested at the bundle

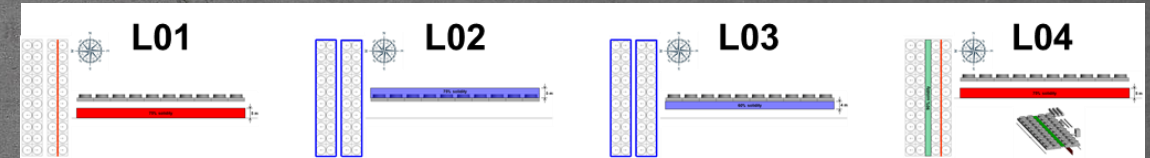
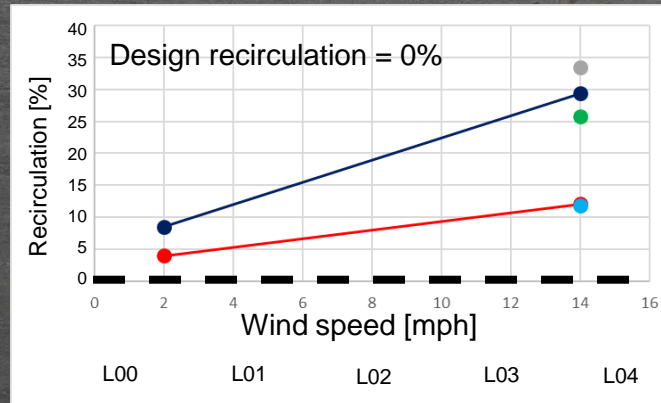
At low wind speed, L00 recirculation is not dramatic but is still significant

L01 (and L04) is able to halve the recirculation with a net gain of -4.54%

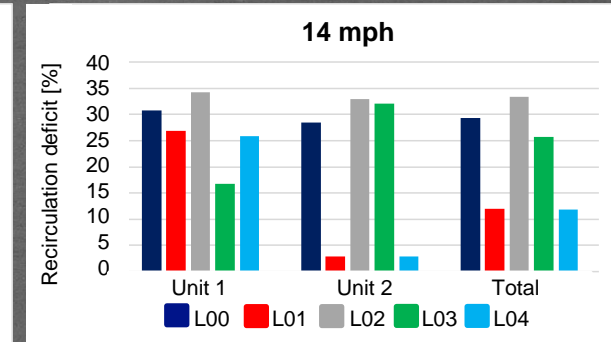
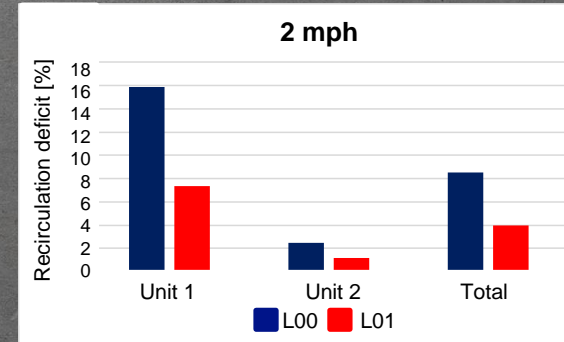
At high wind speed, recirculation without windscreens increase up to 30%

L04 shows at the lowest recirculation: @14 mph the gain is -17.49%

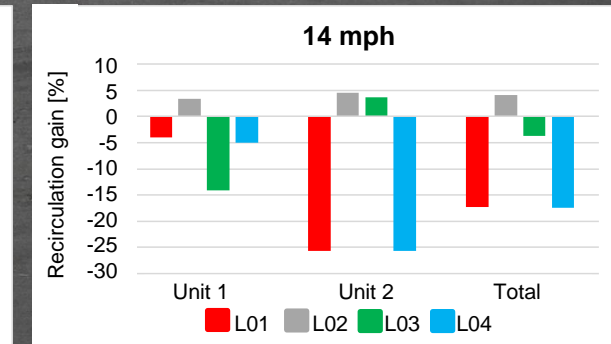
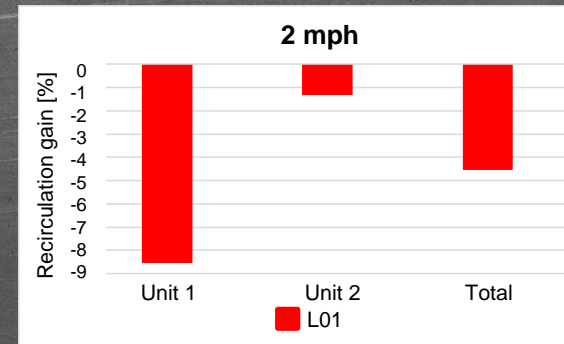
$$R = 100 \cdot \frac{T_{fan} - T_{amb}}{T_{bundle} - T_{amb}}$$



$$R_{deficit} = R - R_{design}$$



$$R_{gain} = R_{L0x} - R_{L00}$$



Baseline results

Thermal power

At low wind speed, thermal power deficit is about 10% mostly due to deficiency on unit 1

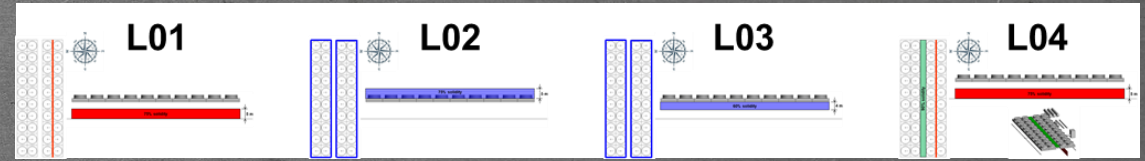
L01 (and L04) is able to further improve the power obtaining a 5.52% gain

At high wind speed, wind wall based screens outperform the other layouts

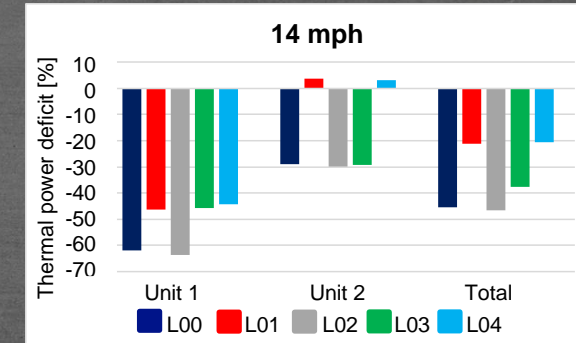
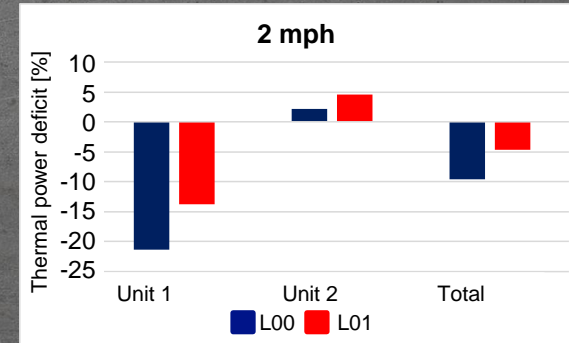
The gain is high on both units with higher benefits on unit 1

L04 shows the best results: @14 mph the gain is 45.42%

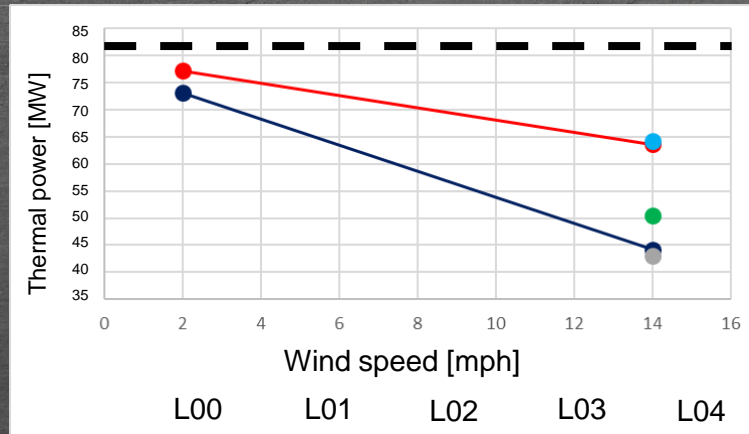
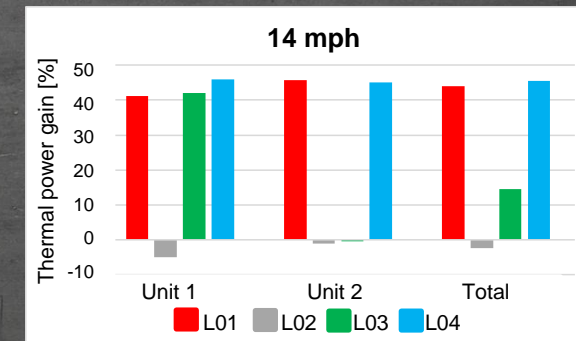
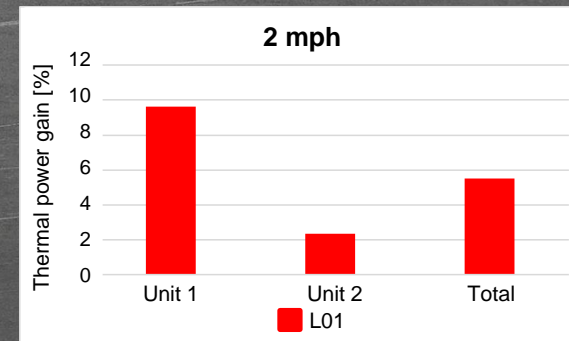
L03 is also showing some improvements



$$\dot{Q}_{deficit} = \frac{\dot{Q} - \dot{Q}_{design}}{\dot{Q}_{design}}$$



$$\dot{Q}_{gain} = \frac{\dot{Q}_{L01x} - \dot{Q}_{L00}}{\dot{Q}_{L00}}$$



Design thermal power = 80.83 MW

Baseline results

Effectiveness

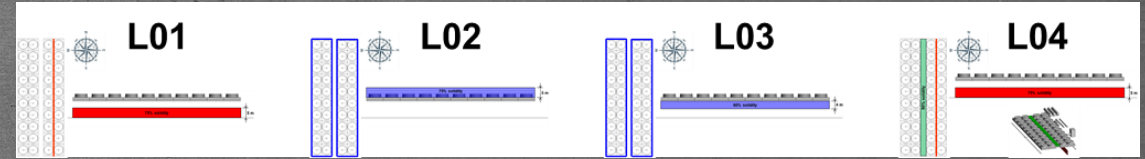
The effectiveness is a measure of the thermal power gained due to the windscreens

Such gain actually corresponds to a reduction of the wind losses

Wind losses with no windscreens are 39.57% @14 mph

At 14 mph such losses are reduced to 12.12% implementing L04

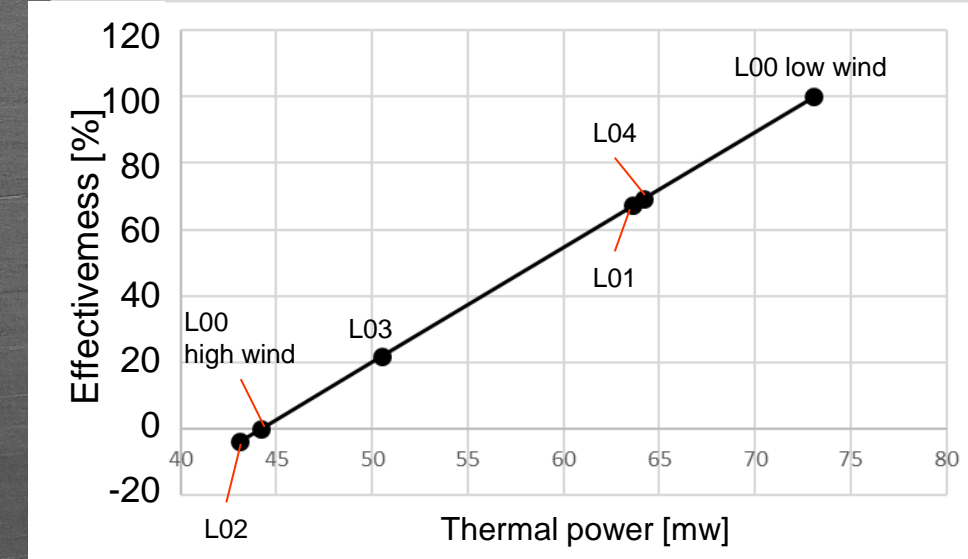
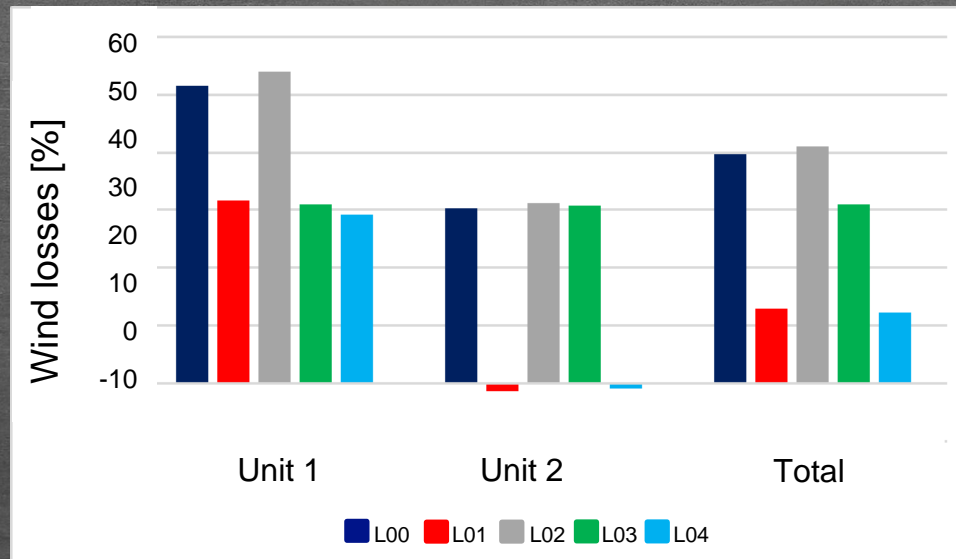
Corresponding effectiveness is **69.37%**



$$\eta_{L0x} = 100 \cdot \frac{\dot{Q}_{L0x \ 14mph} - \dot{Q}_{L00 \ 14mph}}{\dot{Q}_{L00 \ 2mph} - \dot{Q}_{L00 \ 14mph}}$$

Wind losses

$$= \frac{\dot{Q}_{L00 \ 2mph} - \dot{Q}_{L0x \ 14mph}}{\dot{Q}_{L00 \ 2mph}}$$



Conclusions

The implemented CFD modelling is able to accurately reproduce expected cooler performance

At low wind speed, the predicted thermal power is within 10% uncertainty respect to the design spec

Such gap is mainly due to considerable recirculation predicted even at 2 mph

@14 mph wind speed, the wind losses grows up to **40%**

The suggested wind screen layouts are able to mitigate such wind losses significantly

Wind wall based solutions obtain wind losses close to 12%

Corresponding effectiveness is **70%**

Perimeter screens are also able to provide some benefits

Interesting could be to verify the combined benefits of wind wall and perimeter screens

References

- ~ *US Department of Energy- Geothermal Technologies Office*
- ~ *Wikipedia- List of Geothermal power stations*
- ~ *Geothermal Rising*
- ~ *Wind Barriers to Mitigate Wind Effects on Air-Cooled Condensers, Prepared For: The California Energy Commission, Public Interest Energy Research Program , Prepared By:John S. Maulbetsch and Michael N. DiFilippo, Consultant, 2016*
- ~ *Study into the effectiveness of wind screening to mitigate wind effects on ACC fans , Sander Venema, 2015*
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